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Gas-Liquid Flow Regime Maps for Horizontal Pipelines: Predicting Flow Regimes Using Dimensionless Parameter Groups

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ABSTRACT

Flow regime maps are essential to gas-liquid flow applications in many industrial processes to accurately identify the flow regimes before estimating multiphase features. Flow regime classifications were originally based on visual observations of two-phase flow experiments. The observations were mapped on two-dimensional plots (called “flow regime maps”) and the boundaries between regimes determined. Over the years, different coordinates have been proposed for the maps (e.g., superficial velocities and momentum fluxes), in search for parameters that are independent of the given experimental set-up. This paper reports a study on developing new flow regime maps with a broader range of applications by using dimensionless parameter groups as the map coordinates. Various flow regime maps were developed with the use of different combinations of these parameter groups, then they were examined and assessed using datasets from published experimental research and the MultiFlowMet II project for validation. This initial feasibility study develops proof-of-concept flow regime maps that demonstrate the potential of dimensionless parameter groups to more accurately characterise multiphase flow in horizontal pipes, with the optimisation of these maps being considered in future works. The analysis revealed that combinations of the mixture Froude number (Fr_m) versus the ratio of gas superficial velocity to liquid superficial velocity (v_{SG}/v_{SL}), with the liquid phase Froude number (Fr_L) versus the gas phase Froude number (Fr_G) show potential for unambiguous identification and mapping of flow regimes, even for datasets with a wider range of operating conditions.

Keywords: Air-water; flow pattern; dimensionless parameters; transition boundary; two-phase flow; Froude number.

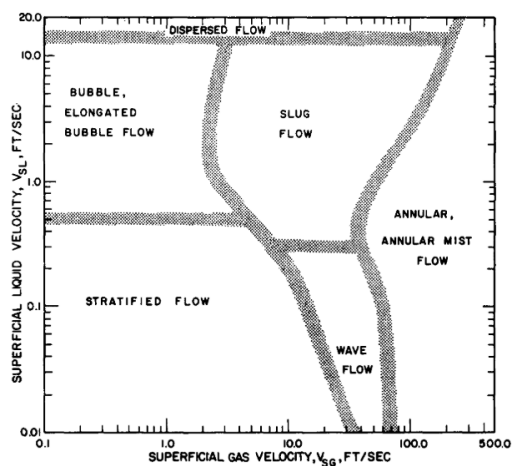
1 INTRODUCTION

Two-phase flows are found in many industrial processes, such as oil and gas production wells and pipelines, heat-exchangers, chemical engineering process plants, nuclear reactors, boiler systems, power plants, geothermal energy, and refrigeration (Humami et al., 2018). In these industries, the predictions of flow characteristics such as pressure-drop and phase fractions are of interest, which are strongly dependent on the prevailing flow regime in the pipes. Therefore, the first step in predicting two-phase flow characteristics is to determine the flow regime (Lips and Meyer, 2011). Different flow regimes can be related to different fluid dynamic mechanisms, so distinct models and correlations could apply only to a specific flow regime (Lips and Meyer, 2011). Flow rates, phases properties, and pipe geometrical characteristics (i.e. equivalent diameter, pipe orientation and roughness) are factors that dictate the occurrence of a particular flow regime (Tzotzi et al., 2011). The community accepts that flow regimes of gas-liquid horizontal flow can be classified as stratified-smooth flow, stratified-wavy flow,

45 slug flow, elongated bubble flow (or plug flow), bubbly flow, disperse bubble flow, annular
46 flow, and wispy annular flow (Amaya-Gomez et al., 2019; França and Lahey, 1992; Zhang et
47 al., 2004). These flow regimes are defined by visual observations mainly, which are subjective
48 (Lin and Hanratty, 1987) and the transition from one flow regime to another identified by
49 various observers can differ.

50 Some discrepancies are observed when data from different sources are brought together and
51 might be due to differences in terminology used by various investigators/authors when the data
52 were collected. Vásquez et al. (2012) categorised bubble flow as dispersed flow, while Govier
53 & Omer (1962) gave bubble flow a distinct class. Usually, transition always exists between
54 two different but neighbouring flow regimes (Kokal and Stanislav, 1989), except for the
55 transition of stratified flow to intermittent flow at low gas velocities (Tzotzi et al., 2011).
56 Shoham (1982) identified a flow configuration as wavy annular flow, which is a subclass of
57 annular flow, while the same flow configuration was considered by Mandhane et al. (1974) as
58 slug flow. Mandhane et al. (1974) grouped bubble flow and plug flow together, as shown in
59 Figure 1 and only separated dispersed bubble flow when the bubbles were uniformly distributed
60 in the liquid phase. Shoham (1982) did not mention bubble flow in his flow regimes
61 classification, but recognised dispersed bubble flow once the elongated bubbles break down
62 into small bubbles. Some researchers only consider a flow configuration to be dispersed bubble
63 flow after the small bubbles have been distributed across the entire pipe (Coleman and
64 Garimella, 1999; Mandhane et al., 1974).

65 Due to the challenges outlined in the previous paragraph, no universally accepted flow regime
66 map for gas-liquid flow in the horizontal pipeline has been established (Baba et al., 2019; Saha,
67 2015). Most developed flow regime maps are dimensional, and only applicable to the
68 experimental datasets used for constructing them (Anumbe, 2018; Brennen, 2005; Cheng et al.,
69 2008). A flow regime map based on dimensionless parameter coordinates would have a broader
70 range of application as the parameters are expected to normalise and delineate all the data
71 points with similar flow characteristics into an area on the map.



72
73 *Figure 1: Gas-liquid flow regime map in horizontal pipe (Mandhane et al., 1974).*

74 A popular approach in dimensionless analysis is the Buckingham Π theorem, where a physical
75 process that satisfies dimensional homogeneity principle and has n relevant variables and m
76 independent dimensions, can be reduced to a relationship between n and m dimensionless
77 parameters (Farokhpoor et al., 2020). However, there is limited success in applying

78 dimensional analysis to gas-liquid flow because it results in many dimensionless parameters
79 and infinite ways in which they can be combined to produce equally valid dimensionless
80 parameter groups (Farokhpoor et al., 2020; Shoham, 2005).

81 Dimensionless parameter groups are formed by incorporating the forces acting on the gas-
82 liquid flow which include inertia, viscosity, gravity (buoyancy) and surface tension (capillary)
83 (Bai and Bai, 2019; Osundare et al., 2020; Zhao and Rezkallah, 1993). The co-current flow of
84 gas and liquid in a horizontal pipe may result in different flow regimes due to various forces
85 acting on the mixture. The balance between these forces dictates the dominating flow regime.

86 At low fluid velocities, the gravity force dominates and generates a tendency for stratification
87 in the vertical direction, i.e. the liquid phase flows at the bottom part of the pipe while the gas
88 phase flows at the top of the pipe. The surface tension force tends to wick the liquid phase
89 around the pipe circumference, thereby creating annular flow; it also tends to form strong
90 interfacial forces that can lead to slug flow (Shi et al., 2017). At high velocity, the inertia force
91 tends to manifest in turbulent fluctuations, where dispersed flows occur. The viscous force
92 promotes annular flow. Therefore, various flow regimes are formed due to the interaction
93 between several forces on the flow components (Bai and Bai, 2019).

94 Various combinations of coordinates for presenting flow regime maps in a 2D graph have been
95 considered in the literature; a chronological overview of developments for generating regime
96 maps for gas-liquid flow in horizontal pipelines is presented in Appendix A. Superficial liquid
97 velocity (v_{SL}) versus superficial gas velocity (v_{SG}) is the most frequently used for constructing
98 flow maps in recent times. This may be due to the assumption that transition curves are least
99 sensitive to changes in pipe diameter and fluid properties when superficial liquid velocity (v_{SL})
100 and superficial gas velocity (v_{SG}) are used as coordinates (Lin and Hanratty, 1987). Al-Sheikh
101 et al. (1970) and Mandhane et al. (1974) developed flow regime map based on the AGA-API
102 data bank, which allowed using a wide range of values for flow parameters and physical
103 properties of gas and liquid in their studies. Some researchers developed their flow regime
104 maps based on data from other studies (Baker, 1954; Jayawardena et al., 1997; Spedding and
105 Nguyen, 1980).

106 Various names are given to the observed flow regimes, and different classifications of flow
107 regimes have been used by many of the researchers. Wong and Yau (1997) and Spedding and
108 Chen (1980) identified 16 and 13 distinguishable flow regimes, respectively. However, all the
109 flow regimes can be grouped in 4 broader classes of stratified, intermittent, annular, and
110 dispersed bubble flow. Some other researchers prefer to work with 6 groups by splitting
111 stratified flow into stratified-smooth and stratified-wavy, and the intermittent flow into plug
112 and slug flow.

113 The clear disparities between existing, dimensional flow regime maps necessitate a new, more
114 transferable approach. In this paper, the authors propose the use of dimensionless parameter
115 groups to develop flow regime maps that are consistent across different flow loops. Section 2
116 outlines the methodology for defining these initial proof-of-concept maps. In Section 3, the
117 resulting maps are presented and validated using data from the MultiFlowMet II project (Elliott
118 et al., 2021). Conclusions and recommendations for multiphase flow practitioners are presented
119 in Section 4.

120 2 METHODOLOGY

121 In this project, two categories of datasets were employed for analysis, those extracted from
 122 experimental works in the literature and those from the MultiFlowMet II project.

123 The dimensional analysis does not account for heat transfer effects, as they do not strongly
 124 influence hydrodynamics in long-distance transportation of oil and gas (Farokhpour et al.,
 125 2020). There are nine relevant independent variables for a fully developed, steady-state,
 126 isothermal gas-liquid flow in a horizontal pipeline. These include superficial gas and liquid
 127 velocities (v_{SG}) and (v_{SL}), gas and liquid densities (ρ_G), and (ρ_L), pipe diameter (d), gas and
 128 liquid viscosities (μ_G) and (μ_L), surface tension (σ), and the acceleration due to gravity (g). The
 129 three fundamental dimensions are mass (M), length (L) and time (T). Based on the Buckingham
 130 Π theorem, the number of system variables minus the number of fundamental variables is equal
 131 to the number of dimensionless groups. Hence, the horizontal gas-liquid flow system will have
 132 eight dimensionless parameter groups, which are (i) density ratio, (ii) square of gas-phase
 133 Froude number, (iii) ratio of superficial liquid velocity to superficial gas velocity, (iv) inverse
 134 of gas-phase Reynolds number, (v) inverse of liquid phase Reynolds number, and (vi) inverse
 135 of gas-phase Weber number.

136 2.1.1 Formation of the dimensionless parameter groups

137 Dimensionless numbers are ratios of any two of the forces (inertia, viscous, gravity and surface
 138 tension) acting on a flow system. However, to know the influence of more than two forces,
 139 selected dimensionless numbers (resulting from dimensional analysis performed on two-phase
 140 horizontal flow) were combined in various ways to form dimensionless parameter groups. The
 141 target was to combine as many as possible forces in a dimensionless parameter group but avoid
 142 having forces in exponential form, as it may be unphysical to interpret exponential force in a
 143 flow system. Among the relevant dimensionless parameter groups identified in this project are:

- 144 i. The ratio of Reynolds number to Eötvös number (Re/Eo), which combines all the forces
 145 acting on the flow system. A high value of Re/Eo would favour a flow system where
 146 inertia and surface tension forces prevail.
- 147 ii. The Froude number (Fr) captures the ratio of inertia to gravity forces.
- 148 iii. The ratio of gravity force to viscous force (G/V), which is derived by dividing the
 149 Reynolds number by the square of the Froude number, so that the inertia force cancels
 150 out. At a high G/V value, gravity force prevails.
- 151 iv. The capillary number (Ca) is the ratio of viscous to surface tension forces.

152
 153 Various combinations of these dimensionless parameter groups governing the gas-liquid flow
 154 are formed by the ratio of various forces acting on the flow. The ratio of inertia force to viscous,
 155 gravity and surface tension forces are Reynolds number, Froude number and Weber number,
 156 respectively. Another valuable dimensionless parameter is the Eötvös number, which is the
 157 ratio of gravity force to surface tension force. The ratio of gravity force (G) to viscous force
 158 (V) is not captured in any of the dimensionless numbers, and hence it is estimated as suggested
 159 by Shi and Yeung (2017). The G/V can also be expressed as the ratio of Reynolds number (Re)
 160 to square of Froude number (Fr^2). Equations 1 to 5 present relevant dimensionless parameter
 161 groups, where forces influencing gas-liquid flow are captured.

$$\frac{Re_i}{Eo} = \frac{\rho_i v_{si}}{\mu_i} \frac{\sigma}{\Delta\rho g d} \quad 1$$

$$Fr_i^2 = \frac{v_i^2 \rho_i}{g d \Delta\rho} \quad 2$$

$$\frac{G}{V} = \frac{\Delta\rho g d^2}{\mu_i v_i} \quad 3$$

$$Ca_i = \frac{\mu_i v_i}{\sigma} \quad 4$$

162 The relevant dimensionless parameter groups Re/Eo , Fr , G/V and Ca stand for the ratio of Reynolds number to
 163 Eötvös number, Froude number, the ratio of gravity to viscous force, and capillary number, respectively. Note:
 164 where, i stands for subscripts m , G and L that is mixture component, gas-phase, and liquid-phase, respectively. The
 165 symbols ρ , $\Delta\rho$, v , d , σ , g , Q and μ denote density, density difference, velocity, pipe diameter, surface tension,
 166 acceleration due to gravity, volumetric flow rate and viscosity, respectively. The v_{SG} and v_{SL} are superficial gas
 167 velocity and superficial liquid velocity, respectively.

168 The volume average method was employed to estimate the mixture density and viscosity, as
 169 shown in Equations 6 and 7. At the same time, the mixture velocity was estimated as the sum
 170 of the superficial velocity of both phases.

$$\rho_m = \rho_L \lambda + \rho_G (1 - \lambda), \quad 5$$

$$\mu_m = \mu_L \lambda + \mu_G (1 - \lambda), \quad 6$$

171 where λ is the volume of liquid fraction, calculated as

$$\lambda = \frac{Q_L}{Q_L + Q_G} = \frac{v_{SL}}{v_{SL} + v_{SG}} \quad 7$$

172 2.2 Criteria for selecting gas-liquid experimental datasets in the literature

173 Not all published experimental gas-liquid flow datasets are fit to be included for analysis in
 174 this project. Two-phase flow characteristics vary widely depending on flow rates, physical
 175 properties of the two phases, and geometrical variables of the pipe (shape, equivalent diameter,
 176 inclination angle, etc.) (Tzotzi et al., 2011). A few of these parameters were used to guide
 177 selection of the appropriate experimental studies for analysis. Table 1 gives the general
 178 requirements for screening experimental datasets available in the public domain for inclusion
 179 into the data bank for this study. The ranges of operating conditions, geometry and fluid
 180 properties for selecting experimental datasets are provided in the ‘Criterion’ column of Table
 181 1, together with corresponding dimensionless parameters Re , We , Fr , G/V , and Eo .

182 Table 1: Factors considered in selecting suitable literature experimental studies for analysis with relevant
 183 dimensionless parameters.

Group of factors	Factor	Criterion	Reasons
Pipe geometry	Pipe inclination	Horizontal (0)	<ul style="list-style-type: none"> • Pipe inclination influences prevailing flow configurations. According to Spedding and Nguyen (1980), some flow regimes appeared only at a particular pipe inclination and not at others, but all the flow regimes that occurred at any angle of inclination also appeared in the horizontal flow condition. • The project is entirely on horizontal pipelines, the influence of dominant forces (inertial, viscous, and gravity) change with pipe angle of inclinations.
	Pipe internal diameter	10-100 mm	<ul style="list-style-type: none"> • The studies on multiphase pipe flows were conducted in closed channels with inner diameters ranging from 10

mm to 100 mm. The multiphase flow behaviours in small diameter pipes (<10 mm) are significantly different from the conventional scales, for example, the surface tension force becomes important, and separated flow is not observed (Fukano and Kariyasaki, 1993; Yang and Shieh, 2001). Likewise, large diameter pipes (>100 mm) have different flow configurations (Kaji and Azzopardi, 2010). Ohnuki and Akimoto (2000) reported that no conventional Taylor bubble appears in large diameter pipe.

- Shao et al. (2009) gave three major differences comparing flows in microchannels and macro systems. In microchannels, (i) the relative importance of surface to volume forces increases; (ii) Laminar flow is usually dominant because of small Reynolds number, and hence viscous forces dominate the inertia forces; (iii) the effects of wettability, wall roughness and flow confinement become important.

Fluid properties	Density	Air $\approx 1.2 \text{ kg/m}^3$, Water $\approx 1000 \text{ kg/m}^3$, Oil $\approx 800-900 \text{ kg/m}^3$.	<ul style="list-style-type: none"> • Researchers' findings support that a high-density difference between phases helps in the formation of stratified flow as compared to a low-density difference. • The gas density strongly affects the transitions within the stratified flow regime (i.e. from smooth to 2-D waves and from 2-D waves to K-H waves) (Tzotzi et al., 2011). • An increase in gas density does not change the transition to slug or plug flow, but significantly decreases the critical gas velocity for the onset interfacial waves and liquid atomization (Hoogendoorn and Buitelaar, 1961; Tzotzi et al., 2011).
	Viscosity	Air $\approx 0.00001 \text{ Pa s}$, Water $\approx 0.001 \text{ Pa s}$, Oil $\approx 0.007 \text{ Pa s}$.	<ul style="list-style-type: none"> • Liquid viscosity does not significantly affect the transition between flow regimes (Hoogendorn, 1959; Weisman et al., 1979). • Matsubara and Naito (2011) reported that when liquid viscosity is higher than 100 mPa s, the transition lines do not agree with previous studies.
	Surface tension	0.03-0.072 N/m	<ul style="list-style-type: none"> • Surface tension affects the transition between flow regimes differently. It is more distinct at low gas and liquid velocities but diminished at high gas and liquid velocities (Bageri et al., 2017).
Flow conditions	Temperature	$\approx 25^\circ\text{C}$	<ul style="list-style-type: none"> • The temperature has strong effects on fluid properties (i.e. viscosity, density, surface tension), which can influence the flow momentum and subsequently the flow configuration.
	Pressure	$\approx 1 \text{ atm}$	<ul style="list-style-type: none"> • The operating pressure affects the flow regimes by the virtue of its influence on the fluid properties (Anumbe, 2018).
Dimensionless parameters	Reynolds number (Re)	300-2300000	<ul style="list-style-type: none"> • It is the ratio of inertia to viscous force in a two-phase flow. A high Re value implies that the flow system is probably an inertia force dominated and turbulent. This type of flow is more likely when the viscosities of the fluids are low, and the pipe diameter and fluids velocities are high.
	Weber number (We)	0.05-200000	<ul style="list-style-type: none"> • This is the ratio of inertia to surface tension force in a two-phase flow. A low We value indicates the dominance of surface tension; in this flow condition, separated flow is unlikely.

Froude number (Fr)	0.01-40	<ul style="list-style-type: none"> • It is the ratio of inertia to gravity force in a two-phase flow. A low Fr value shows that gravity force dominance is more likely.
Gravity to viscous (G/V)	200-9000000	<ul style="list-style-type: none"> • It is the ratio of gravity to viscous force in a two-phase flow. A low G/V value implies a flow system dominated by viscous force, which is more likely with fluids of high viscosity.
Eötvös number (Eo)	10-2000	<ul style="list-style-type: none"> • This is the ratio of gravity to surface tension force in a two-phase flow. It is independent of fluid velocities. It indicates if a flow system is gravity or surface tension force dominated.

2.3 Experimental datasets extracted from literature

All the literature datasets are obtained from experimental gas-liquid flow in horizontal pipes. The literature experimental database contains 2665 records contributed from 17 different experimental set-ups. Air-fresh water flow is 85.4% of the data, which operated at an average temperature of 20 °C, with the fluid properties of 1.204 and 998.3 kg/m³ as air and water density, 0.000018205 and 0.001002 Pa s as air and water viscosity, respectively, and air-water surface tension of 0.072 N/m (Barnea et al., 1980; Chen, 1979; Kong and Kim, 2017; Mathure, 2010; Nguyen, 1975; Shoham, 1982; Spedding et al., 1989; Todkar et al., 2016; Weisman et al., 1979; Wong and Yau, 1997). Light oil and air were used as the working fluids in experimental studies (Gregory et al., 1978; Kokal, 1987; Kokal and Stanislav, 1989) and contributed 14.6% to the database. The density and viscosity of light oil used were 859.9 kg/m³ and 0.0074 Pa s, respectively, and the surface tension of air-light oil was 0.0309 N/m.

Table 2 presents the contribution of each experimental data source to the database, with the corresponding pipe diameter. The sum of data points belonging to each flow regime and total data points contributed by each data source are also shown in Table 2.

Table 2: Contribution of experimental data sources to database and each flow regime.

Data source	Diameter (mm)	ST	SW	SL	PL	BB	AN	DB	Total
Kong and Kim (2017)	38.1	23	17	43	21	14	12	0	130
Weisman et al. (1979)a	51	69	56	29	43	8	44	39	288
Weisman et al. (1979)b	12	0	17	40	36	0	36	29	158
Wong and Yau (1997)	12	14	14	53	13	13	61	0	168
Todkar et al. (2016)	23.9	5	5	0	5	5	5	0	25
Spedding et al. (1989)	93.5	3	61	15	0	0	6	0	85
Nguyen (1975)	45.5	35	52	23	0	12	60	0	182
Chen (1979)	45.5	19	73	12	0	8	85	0	197
Kokal (1987)a; Kokal and Stanislav (1989)a	25.8	1	0	22	13	0	31	12	79
Kokal (1987)b; Kokal and Stanislav (1989)b	51.2	14	0	25	20	0	22	13	94
Kokal (1987)c; Kokal and Stanislav (1989)c	71.3	28	0	10	7	0	3	0	48
Shoham (1982)a	51	24	30	45	22	0	26	21	192
Shoham (1982)b	25	50	25	53	33	0	40	13	214
Gregory et al. (1978)a	25.8	0	0	62	0	0	0	0	62
Gregory et al. (1978)b	51.2	0	0	105	0	0	0	0	105
Barnea et al. (1980)	25	37	19	40	27	0	32	15	170
Mathure (2006)	12.7	0	0	302	118	22	26	0	468
Total		346	369	879	358	82	489	142	2665

Note: ST means stratified-smooth, SW means stratified-wavy, SL means slug, PL means plug, AN means annular, BB means bubble, and DB means dispersed bubble flow.

202 The major difficulties in compiling these data from the experimental research works are the
 203 ambiguity in the definition and different nomenclatures adopted by various researchers to
 204 describe the flow regimes (Sharma et al., 2006). Thus, a more objective and consistent way
 205 was adopted by critically analysing the definitions given to each flow regime by a researcher
 206 and then assigning the class where the flow regime fits. For sizeable and meaningful analysis,
 207 the flow regimes with common features were grouped to form stratified-smooth (ST),
 208 stratified-wavy (SW), slug (SL), plug (PL), annular (AN), bubble (BB), and dispersed bubble
 209 (DB) flow.

210 In all the considered experimental works for data extraction, both bubble flow and dispersed
 211 bubble flow were not considered individually in the same experiment. It is either bubble flow
 212 being considered, or dispersed bubble flow being considered, but not both. Weisman et al.
 213 (1979)a is the only exception, where construction of a gas-liquid flow map for 51mm pipe
 214 diameter contained both bubble flow and dispersed bubble flow. Therefore, in this project,
 215 datasets for both bubble flow and dispersed bubble flow are combined and considered to be a
 216 single flow regime. These two flow regimes will be referred to as dispersed bubble (DB) flow
 217 in the rest of this paper.

218 **2.4 The MultiFlowMet II experimental setup**

219 As well as utilising data acquired from literature survey, this study will also include several
 220 data sets generated from an experimental study as part of the European Metrology Programme
 221 for Innovation and Research (EMPIR) project 16ENG07–MultiFlowMetII. This project
 222 focused on the minimisation of laboratory-specific effects and included an intercomparison
 223 between three leading multiphase flow laboratories (Elliott et al., 2021): TÜV SÜD National
 224 Engineering Laboratory (NEL) in the UK; DNV GL in the Netherlands; and NORCE in
 225 Norway. Across all three facilities, a 76.2 mm (3 in) internal diameter steel pipe was used and
 226 in all test points, a horizontal (0° inclination) configuration was employed. Additional
 227 experimental studies were conducted at the DNV GL facility with internal diameters of 101.6
 228 mm (4 in) and 152.4 mm (6 in). An overview of the variations between the laboratories, and
 229 the physical parameters achieved, are presented in Table 3.

230 Table 3: Summary of the flow parameters and flow loop geometry for the three multiphase laboratories.

Characteristic	NEL, low pressure	NEL, high pressure	NORCE	DNV GL
Gas density (kg/m ³)	5.54-6.41	9.74-11.21	6.26-7.47	8.52-12.03
Gas viscosity (Pa.s)*10 ⁻⁵	1.87-1.88	1.77-1.89	1.80	1.76-1.79
Water density (kg/m ³)	1023.65-1025.49	1022.79-1025.89	1031.58-1032.07	1030.87-1032.03
Oil density (kg/m ³)	814.56-829.60	813.91-818.15	823.52-824.78	825.47-828.50
Oil viscosity (Pa.s)*10 ⁻³	7.26-8.26	6.95-8.51	2.4-2.7	4.8-5.2
Closed/open loop	Open	Open	Closed	Closed
Injection type	Gas into liquid	Gas into liquid	Gas into liquid	Liquid into gas

231 Given that the aim of this research was to assess the influence of laboratory effects on the flow
 232 measurement, the same matrix of test points was repeated at each facility, though this was
 233 separated into a lower pressure (557.28 kPa) comparison between NEL and NORCE, and a
 234 higher pressure (1013.25 kPa) comparison between NEL and DNV GL. The two test matrices
 235 showing the input liquid volumetric flow rate (Q_{liq}) and the corresponding gas volume fraction
 236 (GVF) at lower and higher pressures are presented in Table 4. The GVF value was calculated

237 by gas volumetric flow rate against the total gas and liquid mixture volumetric flow rate local
 238 to the flow regime observation section.

239 Table 4: Multiphase intercomparison test matrices.

Q _{liq} (m ³ /s) *10 ⁻³	<i>Lower pressure</i>							<i>Higher pressure</i>						
	GVF (%)		GVF (%)					GVF (%)		GVF (%)				
	10	30	50	67	80	90	95	10	30	50	67	80	90	95
1.38							X							X
4.17					X	X						X	X	
8.33			X	X	X					X	X	X		
13.89		X	X	X					X	X	X			
20.83	X	X	X	X				X	X					
27.78	X													

240 The facilities are 3-phase systems, but the focus of this project is 2-phase flow. Therefore, the
 241 oil and water components were combined to form the liquid phase. So, the data points from the
 242 facilities can be used/employed for gas-liquid analysis. The MultiFlowMet II project primarily
 243 focused on intermittent flow, hence it generated only a few data points for stratified wavy,
 244 annular and dispersed bubble flows, with no records of the other flow regimes, as presented
 245 in Table 5.

246 Table 5: Data from the MultiFlowMet II project.

Diameter (mm)	Operating pressure (kPa)	Stratified wavy	Intermittent	Annular	Dispersed bubble	Total
101.6	101.32	0	326	5	2	333
152.4	101.32	1	321	3	0	325
76.2	1013.25	0	79	8	0	87
76.2	557.28	0	37	8	0	45
Total		1	763	24	2	790

247 Note: Intermittent flow is a combination of slug flow and plug flow.

248 A total of 3455 datasets are available for analysis, 790 of these are from MultiFlowMet II and
 249 the remainder is extracted from experimental projects in scholarly journal articles. The major
 250 differences between the experimental datasets extracted from literature and those from the
 251 MultiFlowMet II project are as follows: most of the experiments from the literature were
 252 conducted at standard temperature and pressure conditions, and the datasets are all for pipe
 253 diameters between 10-100 mm; in MultiFlowMetII, the datasets were recently generated, with
 254 83.8% of the data points obtained in pipe diameters greater than 100 mm at standard conditions,
 255 42.2% out of this 83.8% were obtained with a pipe diameter of 101.6 mm, and 16.7% of the
 256 data points obtained with a pipe diameter of 76.2 mm were at elevated operating pressures
 257 (1013.25 kPa and 557.28 kPa). Therefore, the MultiFlowMet II datasets were employed to
 258 validate the constructed flow maps and check the applicability of constructed flow maps
 259 outside the range of the original datasets used for their construction.

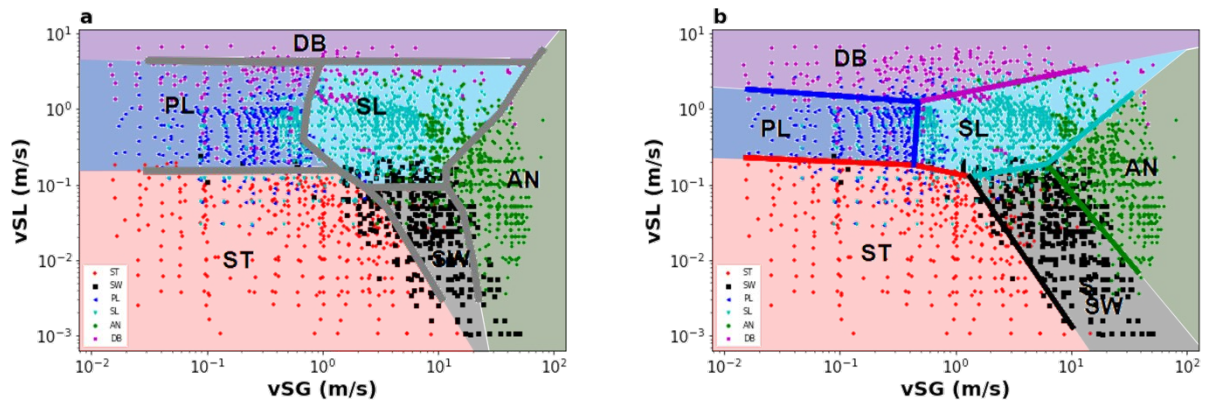
260 3 RESULTS AND DISCUSSION

261 3.1 Construction of flow regime maps

262 All experimental datasets extracted from the literature are presented on the Mandhane et al.
 263 (1974) flow regime map in Figure 2a, which utilises superficial liquid velocity (v_{SL}) and
 264 superficial gas velocity (v_{SG}) as the axes. There are no newly-drawn transition boundaries in
 265 Figure 2a *Figure 2*; the Mandhane et al. (1974) transition boundaries were superimposed on data
 266 points gathered from the literature. The literature data seem to fit with the superimposed

267 Mandhane et al. (1974) transition boundaries in some cases, though there are notable areas that
 268 do not show agreement. The challenge in predicting transition boundaries can be easily
 269 observed in Figure 2a, particularly at the transition between SL/SW, SL/AN and the other flow
 270 regimes. As will be discussed below, only 66.6% of the data points were correctly characterised
 271 by the standard flow regime map. Therefore, there is scope for significant improvement in the
 272 predictive capabilities. Figure 2b presents a flow regime map with the coordinates of superficial
 273 liquid velocity (v_{SL}) versus superficial gas velocity (v_{SG}) with transition boundaries drawn to
 274 mark each flow regime to a distinct region. The authors note that these transition boundaries
 275 represent a non-unique classification designed to demonstrate possible improvements in the
 276 number of correctly classified test points without the utilisation of statistical methods to provide
 277 an optimised categorisation. Based on the constructed transition boundaries, 75.7% of the
 278 analysed data points were predicted correctly. In the remainder of this section, dimensionless
 279 parameter groups are used to investigate flow regime maps with more distinct boundaries.

280

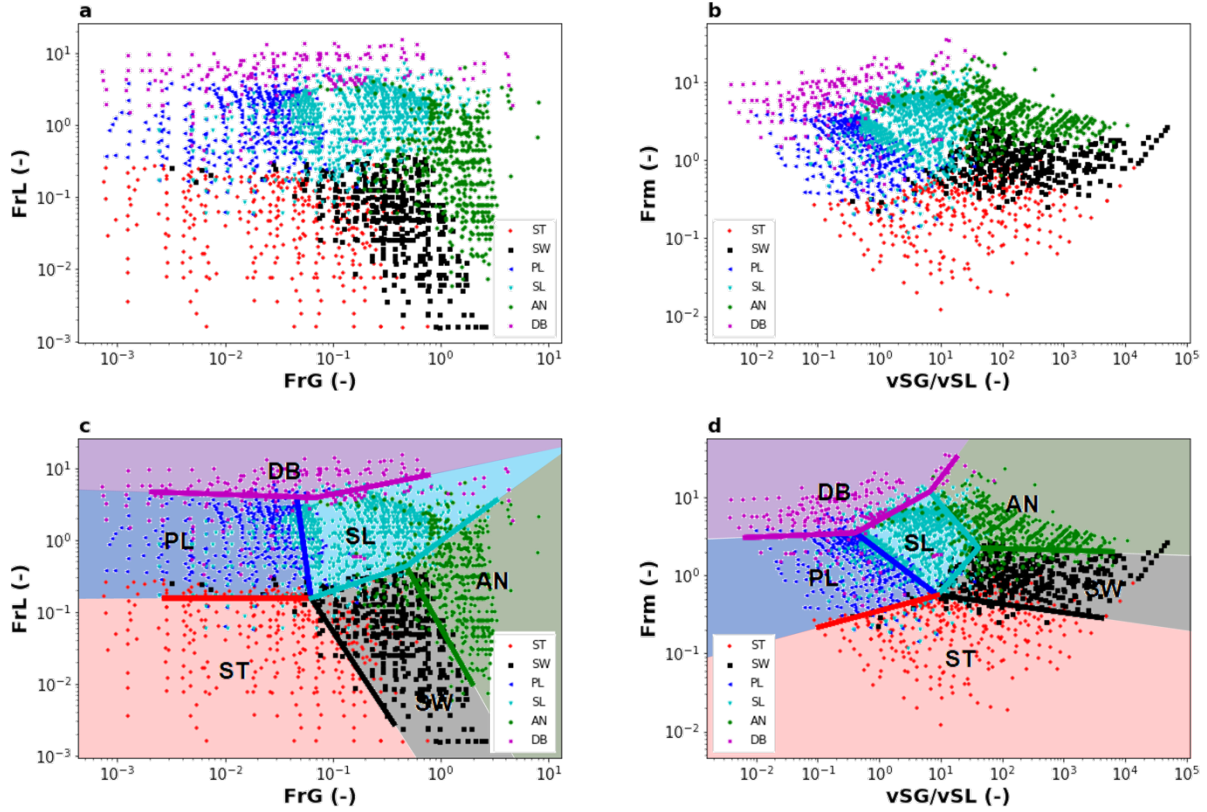


281

282 *Figure 2: The flow map of superficial liquid velocity (v_{SL}) versus superficial gas velocity (v_{SG}) based on gathered*
 283 *literature datasets, (a) with superimposed Mandhane et al. (1974) flow map, (b) with constructed*
 284 *transition boundaries. Several combinations of dimensionless parameter groups were employed for*
 285 *constructing flow regime maps. Some of the dimensionless parameter groups were used to*
 286 *normalise the chosen datasets, which were split into those from MultiFlowMet II and those*
 287 *extracted from published experimental works. For MultiFlowMetII, the intermittent flow was*
 288 *the focus, which was a combination of plug and slug flows, so the analysis of this grouping*
 289 *considered both plug and slug flows as intermittent flow. In the construction of flow regime*
 290 *maps, there are inherent challenges due to data points from different research works being*
 291 *assigned different names for a given flow regime, and the use of visual observation to identify*
 292 *flow regimes in a subjective manner. Moreover, due to dependence on the active physical*
 293 *mechanisms, it is unlikely that any set of two dimensionless parameters will provide a unified*
 294 *description of all flow regime transitions.*

295 To address this challenge, this work considers ratios of dimensionless parameters to assess their
 296 suitability for providing a more unified flow regime map. Those combinations that neither
 297 normalised nor delineated the flow regimes will not be discussed here. Some combinations of
 298 dimensionless parameter groups give the same type of information; this is the case of We_m/Eo
 299 versus v_{SG}/v_{SL} and F_{rm} versus v_{SG}/v_{SL} , as well as We_L/Eo versus We_G/Eo and Fr_L versus Fr_G .
 300 The reason being that, by definition, both We/Eo and Fr are ratios of inertial to gravity forces,
 301 and Fr is the square root of We/Eo value (Osundare et al., 2020).

302 A total of seven combinations of dimensionless parameter groups was found to show good
 303 potential for constructing flow maps, however the best two flow maps are presented in Figure
 304 3a to Figure 3b. Transition boundaries were drawn on these flow maps to enclose data points
 305 from a flow regime that could be clearly delineated into a distinct area or region on the flow
 306 map. No transition boundaries were drawn in the absence of a clear distinction between data
 307 points of two flow regimes.



308
 309 *Figure 3: Constructing transition boundaries for datasets extracted from experimental works in the literature; (a)*
 310 *Fr_L versus Fr_G ; (b) Fr_m versus v_{SG}/v_{SL} (c) Fr_L versus Fr_G with transition boundaries; (d) Fr_m versus v_{SG}/v_{SL}*
 311 *with transition boundaries.*

312 All the flow regimes were normalised and delineated by the combination of Fr_L versus Fr_G , as
 313 presented in Figure 3a. Based on the data points analysed, the combination of Fr_m versus
 314 v_{SG}/v_{SL} normalised and delineated all the considered flow regimes, as presented in Figure 3b.
 315 The transition boundaries associated with the maps of Fr_L versus Fr_G and Fr_m versus v_{SG}/v_{SL}
 316 are presented in Figure 3c and Figure 3d, respectively.

317 3.2 Performance of different combinations of dimensionless parameter groups

318 The accuracy of various combinations of dimensionless parameter groups was estimated by
 319 verifying the data points from experimental works in the literature against the constructed
 320 transition boundaries. The data points are grouped into matched, nearly matched, and miss-
 321 matched. The matched data points are those that were normalised and delineated and captured
 322 within the constructed transition boundaries of a particular flow regime. The nearly matched
 323 data points are those that were delineated but not captured within the constructed transition
 324 boundaries of that particular flow regime. The miss-matched data points are those that were
 325 neither normalised nor delineated and not near to the constructed transition boundaries of a
 326 specific flow regime. Data points enclosed by the transition lines were counted as matched.
 327 Parallel lines were drawn at some distance to the transition boundaries considering those data

328 points that were delineated but not captured within the constructed transition boundaries, and
 329 the data points inside these lines were counted as nearly-matched, and those outsides were
 330 counted missed-matched. Table 6 presents a summary of the performance of various
 331 combinations of dimensionless parameter groups. The values in Table 6 are percentages of data
 332 points of a particular flow regime that are either matched or nearly matched or miss-matched.

333 Table 6: Accuracy of various combinations of dimensionless parameters and Mandhane et al. (1974) flow map
 334 in predicting some flow regimes in percentage (%).

Flow regimes		ST	SW	SL	PL	DB	AN	Average
v_{SL} versus v_{SG} superimposed by Mandhane et al. (1974) flow map	Matched	88.7	53.1	81.7	88.5	18.8	68.5	66.6
	Nearly matched	11.3	44.7	10.7	9.8	72.3	28.2	29.5
	Miss-matched	0.0	2.2	7.6	1.7	8.9	3.3	3.9
v_{SL} versus v_{SG} with constructed transition boundaries	Matched	84.4	78.3	82.0	57.0	75.4	77.3	75.7
	Nearly matched	15.6	20.1	15.4	39.1	18.3	19.2	21.3
	Miss-matched	0.0	1.6	2.6	3.9	6.3	3.5	3.0
Re_m/Eo versus v_{SG}/v_{SL}	Matched	0.0	80.2	0.0	0.0	53.1	73.2	34.4
	Nearly matched	98.6	17.9	81.2	88.8	25.0	24.1	55.9
	Miss-matched	1.4	1.9	18.8	11.2	21.9	2.7	9.6
Re_L/Eo versus Re_G/Eo	Matched	0.0	79.9	0.0	0.0	53.1	72.8	34.3
	Nearly matched	97.7	18.2	84.9	88.8	25.4	24.3	56.6
	Miss-matched	2.3	1.9	15.1	11.2	21.4	2.9	9.1
Fr_L versus Fr_G	Matched	76.0	81.3	82.5	85.2	63.4	80.16	78.1
	Nearly matched	24.0	16.8	17.5	12.6	27.7	16.16	19.1
	Miss-matched	0.0	1.9	0.0	2.2	8.9	3.68	2.8
Fr_m versus v_{SG}/v_{SL}	Matched	81.2	82.9	79.1	82.1	64.7	81.2	78.5
	Nearly matched	18.8	15.2	20.9	12.8	26.3	15.7	18.3
	Miss-matched	0.0	1.9	0.0	5.0	8.9	3.1	3.2
G/V versus v_{SG}/v_{SL}	Matched	52.9	69.9	82.9	77.4	57.6	87.5	71.4
	Nearly matched	39.0	19.2	15.4	17.9	22.3	7.6	20.2
	Miss-matched	8.1	10.8	1.7	4.7	20.1	4.9	8.4
Ca_L versus Ca_G	Matched	78.9	84.8	0.0	68.4	0.0	81.39	52.3
	Nearly matched	9.0	13.6	100.0	30.4	89.3	16.16	43.1
	Miss-matched	12.1	1.6	0.0	1.1	10.7	2.45	4.7
Ca_m versus v_{SG}/v_{SL}	Matched	79.8	83.7	0.0	65.4	0.0	82.62	51.9
	Nearly matched	8.1	14.6	100.0	32.7	89.3	14.93	43.3
	Miss-matched	12.1	1.6	0.0	2.0	10.7	2.45	4.8

335 Note: ST = stratified-smooth, SW = stratified-wavy, SL = slug, PL = plug, AN = annular, and DB = dispersed
 336 bubble flow.

337 The v_{SL} versus v_{SG} data was demonstrated to match 66.6% of the test points when the classical
 338 Mandhane et al. (1974) flow map was applied, but this figure was increased to 75.7% when the
 339 approach outlined in this paper was applied. This increase confirms the hypothesis that there is
 340 potential to improve the classification of flow regimes by using dimensionless parameter
 341 groups, though the authors note that maps presented in this paper provide a proof-of-concept

342 for such improvements, rather than an optimal categorisation. Moreover, the combinations of
343 Fr_m versus v_{SG}/v_{SL} (78.5% of points matched) and Fr_m versus Fr_L (78.1% of points matched)
344 demonstrate the potential for further increases in accuracy through the exploration of additional
345 dimensionless parameter groups.

346 For the ST flow regime, the combinations Fr_m versus v_{SG}/v_{SL} , Fr_L versus Fr_G , and the v_{SL} versus
347 v_{SG} superimposed by Mandhane et al. (1974) have flow map zero miss-matched cases. All the
348 combinations of dimensionless parameter groups have a matched result $>75\%$ except for G/V
349 versus v_{SG}/v_{SL} with 52.9%, as shown in Table 6. The v_{SL} versus v_{SG} superimposed by Mandhane
350 et al. (1974) has the highest matched cases of 88.7%.

351 For the SW flow regime, the v_{SL} versus v_{SG} superimposed by Mandhane et al. (1974) has the
352 least matched cases of 53%, all the combinations of dimensionless parameter groups have a
353 matched result of $>80\%$, with the exception of G/V versus v_{SG}/v_{SL} with 69.9%, as shown in
354 Table 6. The miss-matched result in all the combinations of dimensionless parameter groups is
355 1.9%, which is lesser than 2.2% miss-matched result for the superimposed Mandhane et al.
356 (1974) flow map except for G/V versus v_{SG}/v_{SL} with 10.8%, as shown in Table 6.

357 For the SL flow regime, the combinations of Fr_m versus v_{SG}/v_{SL} and Fr_L versus Fr_G have zero
358 miss-matched cases. These combinations of dimensionless parameter groups have a $>80\%$
359 matched result, with the exception of Fr_m versus v_{SG}/v_{SL} with 79.1%, whereas the superimposed
360 Mandhane et al. (1974) flow map has a matched result of 81.7% with a miss-matched result of
361 7.6%, as shown in Table 6. Both Re_m/Eo versus v_{SG}/v_{SL} and Re_L/Eo versus Re_G/Eo flow maps
362 have zero matched results because no region can be marked out for the slug flow. In Ca_L versus
363 Ca_G and Ca_m versus v_{SG}/v_{SL} flow maps, zero matched results were recorded because the bubble
364 flow data points overlapped the slug region, but the slug flow data points were all normalised
365 giving a miss-matched and nearly matched results of 0% and 100%, respectively.

366 For the PL flow regime, both Re_m/Eo versus v_{SG}/v_{SL} and Re_L/Eo versus Re_G/Eo flow maps have
367 zero matched results. The superimposed Mandhane et al. (1974) flow map and all the other
368 combinations of dimensionless parameter groups in exception of Ca_L versus Ca_G and Ca_m
369 versus v_{SG}/v_{SL} flow maps recorded a $>75\%$ matched result, as presented in Table 6.

370 For the DB flow regime, apart from Ca_L versus Ca_G and Ca_m versus v_{SG}/v_{SL} flow maps where
371 the DB data points formed more than one trend and no matched result could be recorded, v_{SL}
372 versus v_{SG} with constructed transition boundaries has the highest matched result of 75.4%, but
373 the superimposed Mandhane et al. (1974) flow map has the lowest matched result (18.8%), as
374 presented in Table 6. The combination of Re_m/Eo versus v_{SG}/v_{SL} has the highest miss-matched
375 result of 21.9%, as presented in Table 6. The DB flow regime has the highest miss-matched
376 result which could be due to the DB flow data being a combination of data from two distinct
377 flow regimes (bubble flow and disperse bubble flow).

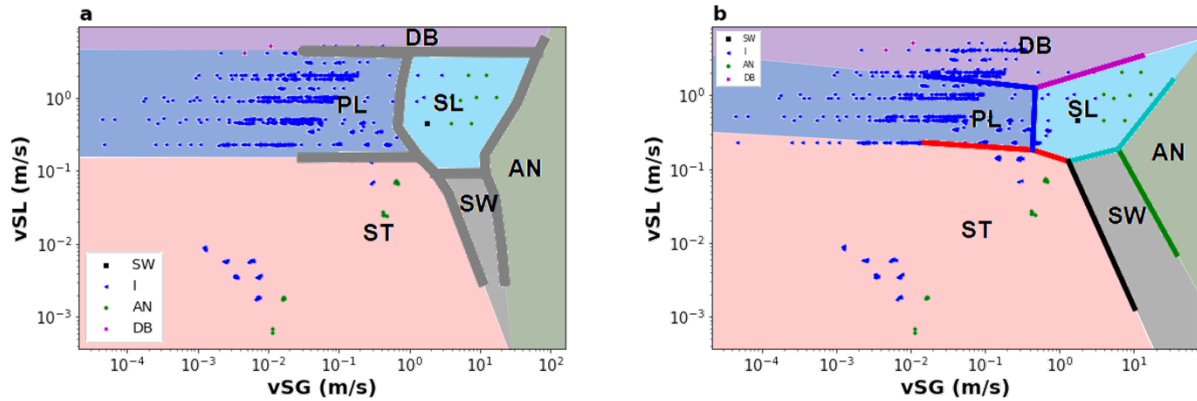
378 For the AN flow regime, the matched result recorded in all the combinations of dimensionless
379 parameter groups is better than 68.5% for the superimposed Mandhane et al. (1974) flow map.
380 The combination of G/V versus v_{SG}/v_{SL} has the highest matched result of 87.5%, but also with
381 the highest miss-matched result (4.9%).

382 3.3 Superimposing the transition boundaries on the MultiFlowMet II datasets

383 3.3.1 Transition boundaries superimposed on superficial liquid velocity versus 384 superficial gas velocity

385 Figure 4 presents the flow regime map of the MultiFlowMet II datasets using the y-x axes of
386 the superficial liquid velocity (v_{SL}) versus superficial gas velocity (v_{SG}) with superimposed
387 transition boundaries.

388



389

390 *Figure 4: The flow map of superficial liquid velocity (v_{SL}) versus superficial gas velocity (v_{SG}) based on the*
391 *datasets from MultiFlowMet II projects, (a) superimposed Mandhane et al. (1974) flow map, (b) with constructed*
392 *transition boundaries.*

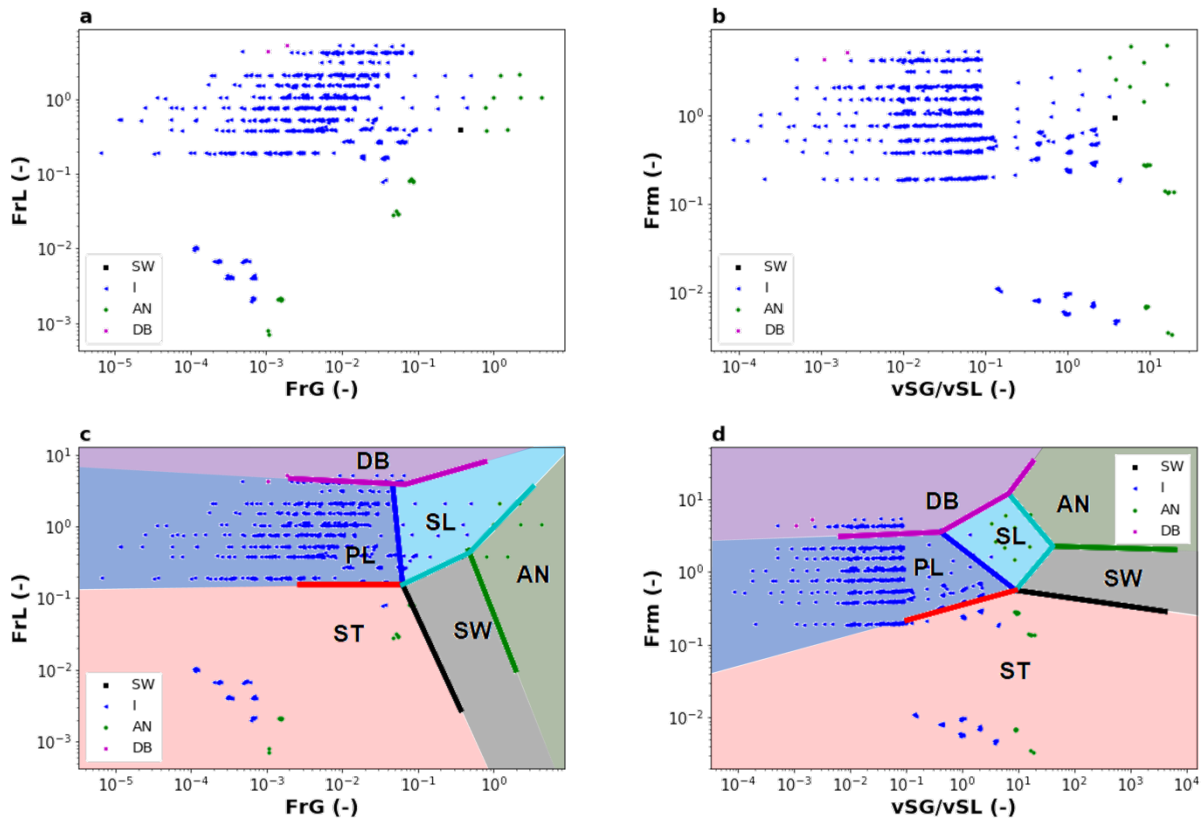
393 Note: Intermittent flow (I) is a combination of slug flow (SL) and plug flow (PL).

394 The MultiFlowMet II data points are presented with the Mandhane et al. (1974) and constructed
395 flow maps in Figure 4a and 4b, respectively. While the classical map correctly characterises
396 95.5% of the test points, only 67.9% of points are correctly identified by the constructed flow
397 regime map. In particular, a significant number of PL data points are incorrectly categorised as
398 DB, though the two dispersed bubble flow points are more accurately categorised. This
399 disparity is likely explained by the fact that 321 of the 763 intermittent test points were
400 conducted using a pipe diameter of 152.4 mm, which is beyond the 10-100 mm range used to
401 define the flow boundaries, which was not the case for Mandhane et al. (1974). This distinction
402 raises an important point regarding the scalability of flow regime maps, as will be explored in
403 future work.

404 3.3.2 Dimensionless parameter group transition boundaries

405 As can be concluded from Section 3.3.1, there is clear scope to improve the accuracy of the
406 newly constructed flow regime maps through the application of alternative dimensionless
407 parameter groups. The combinations listed in Table 6 were again utilised for the MultiFlowMet
408 II data. From the seven combinations of dimensionless parameter groups analysed, Fr_L versus
409 Fr_G and F_{rm} versus v_{SG}/v_{SL} flow maps are presented in Figure 5; these results represent the
410 most viable candidates for delineating the new data set.

411



413

414

415 *Figure 5: Superimposing the transition boundaries of datasets extracted from experimental works in the literature*
 416 *on the datasets from MultiFlowMet II projects; (a) Fr_L versus Fr_G ; (b) Fr_m versus v_{SG}/v_{SL} . (c) Fr_L versus Fr_G with*

416

417 The Fr_L versus Fr_G flow regime map, presented in Figure 5c, correctly matched 92.7% of
 418 intermittent flow test points, whereas a figure of 88.9% was achieved for the Fr_m versus v_{SG}/v_{SL}
 419 map. Thus, this new choice of parameter groups represents a clear improvement from the
 420 attempts made using only the superficial velocities. This is likely to be as a result of the explicit
 421 inclusion of the pipe diameter in the definition of Froude number (Equation 2). However, the
 422 slight underperformance in comparison with the traditional flow regime map suggests that
 423 further improvements could be made by including physical knowledge of the flow regime
 424 transitions in constructing the boundaries.

425 The incorrect prediction of annular test points is largely consistent between the v_{SG} versus v_{SL}
 426 maps (traditional and constructed) and the Fr_m versus v_{SG}/v_{SL} map, though this is somewhat
 427 improved by the comparison of Froude numbers, which correctly identifies 25% of the points.
 428 Of the dimensionless parameter groups considered, no significant improvement on this fraction
 429 was achieved, which may be explained by the impact of variations in pipe diameter on the
 430 transition from stratified flow (Emamzadeh and Issa, 2013).

431 3.4 Discussion of results

432 The combinations of dimensionless parameter groups employed for constructing flow regime
 433 maps in this project are:

- 434 1. The ratio of mixture Reynolds number to Eötvös number (Re_m/Eo) versus the ratio gas
 435 superficial velocity to liquid superficial velocity (v_{SG}/v_{SL}).

- 436 2. The ratio of liquid Reynolds number to Eötvös number (Re_L/Eo) versus gas Reynolds
437 number to Eötvös number (Re_G/Eo).
- 438 3. The mixture Froude number (Fr_m) versus the ratio gas superficial velocity to liquid
439 superficial velocity (v_{SG}/v_{SL}).
- 440 4. The liquid Froude number (Fr_L) versus gas Froude number (Fr_G).
- 441 5. The liquid capillary number (Ca_L) versus gas capillary number (Ca_G).
- 442 6. The mixture capillary number (Ca_m) versus the ratio gas superficial velocity to liquid
443 superficial velocity (v_{SG}/v_{SL}).
- 444 7. The ratio of gravity force to viscous force (G/V) versus the ratio gas superficial velocity
445 to liquid superficial velocity (v_{SG}/v_{SL}).

446 Some combinations of dimensionless parameter groups such as Fr_L versus Fr_G , and Fr_m versus
447 v_{SG}/v_{SL} were successful in normalising and delineating the MultiFlowMet II intermittent data
448 points and were able to match the superimposed transition boundaries by 92.7% and 88.9%,
449 respectively, as presented in Figure 5c and 5d. This was in spite of a wide range of pipe
450 diameters (76.2-152.4 mm) and 3 different operating pressures (101.32, 557.28 and 1013.25
451 kPa.) used in MultiFlowMetII. These combinations of dimensionless parameter groups have
452 good potential to normalise and delineate intermittent flow.

453 Clerc (2000) and De Lorenzo et al. (2017) stated that the mixture-component based on
454 homogeneous equilibrium model is well-designed to simulate/predict the dispersed bubble
455 flow, but cannot reproduce mechanical or thermodynamic non-equilibrium flow, such as
456 annular flows. This is consistent with the results obtained when superimposing the transition
457 boundaries from experimental works in the literature on MultiFlowMet II datasets. Only 4.2%
458 of MultiFlowMet II annular data points fit in the annular region of the superimposed transition
459 boundaries for Fr_m versus v_{SG}/v_{SL} , but the combinations of Fr_L versus Fr_G has 25%, as shown
460 in Figure 5d and Figure 5c. Yet, the superimposed Mandhane et al. (1974) flow map has zero
461 matched results for the MultiFlowMet II annular data points.

462 In all flow maps constructed and analysed using the MultiFlowMet II intermittent data points,
463 2.4% of these data points were miss-matched. These miss-matched data points appeared to
464 have a separate trend from the rest of the data points. All the data points in this category were
465 found to be from the same source, the MultiFlowMet II experimental set-up with a pipe
466 diameter of 76.2 mm and an operating pressure of 557.28 kPa. The annular flow data points in
467 the same region were also miss-matched and from the same MultiFlowMet II experimental set-
468 up. Once more, the disparity of the pressure—557.28 kPa for these test points in comparison
469 with the value of 1 atm (101.325 kPa) for the flow map definition—has a clear impact on their
470 accuracy. Future developments in this area should account for such variations to create a more
471 scalable characterisation.

472 The flow regime maps for We_m/Eo versus v_{SG}/v_{SL} and Fr_m versus v_{SG}/v_{SL} gave the same type
473 of information, except for the magnitude of values on the vertical axis, as Fr^2 equals We/Eo .
474 Similar observations apply to the flow maps of We_L/Eo versus We_G/Eo and Fr_L versus Fr_G
475 although in this case the magnitude of values differs for both abscissa and ordinate in the flow
476 maps of We_L/Eo versus We_G/Eo and Fr_L versus Fr_G . Therefore, data points with the same flow
477 characteristics would be normalised and delineated into a similar flow regime when axes of the
478 flow map capture the same ratio of forces influencing a flow system.

479 **4 CONCLUSIONS**

480 In the quest to construct more generalised flow regime maps for gas-liquid flow in horizontal
 481 pipelines, datasets were obtained from the MultiFlowMet II project and relevant experimental
 482 studies in the literature, to create a robust databank for analysis. Dimensional analysis was
 483 performed on the gas-liquid flow in a horizontal pipe to obtain relevant dimensionless
 484 parameter groups, combining forces acting on the flow system.

485 Based on the analysed datasets, the combinations of dimensionless parameter groups
 486 formulated using mixture-component is less effective than those developed using the phase-
 487 component in normalising and delineating annular flow. This is because combinations based
 488 on mixture-components are well-designed to simulate mechanical or thermodynamic
 489 equilibrium flows, such as dispersed bubble flow, but cannot reproduce mechanical or
 490 thermodynamic non-equilibrium flows like annular flow.

491 Based on the analysed datasets, the combinations of the Fr_m versus v_{SG}/v_{SL} and Fr_L versus Fr_G
 492 dimensionless parameter groups show good potential in normalising and delineating the
 493 MultiFlowMet II intermittent data points. Therefore, the feasibility study presented in this work
 494 provides important evidence for improving the prediction of multiphase flow regimes in
 495 horizontal pipes through the use of dimensionless parameter groups. The development of this
 496 work will focus on the introduction of a physics-informed machine learning approach to
 497 produce optimised flow regime maps that incorporate physical knowledge of individual flow
 498 regime transitions.

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 504 Horizon 2020 research and innovation programme and the EMPIR participating states.

505 **CONFLICTS OF INTEREST**

506 The authors declare no conflict of interest.

507 **NOMENCLATURE**

Symbol/ Abbreviation	Unit	Interpretation	Flow regime Abbreviation	Interpretation
A	m ²	Pipe cross-sectional area	AN	Annular flow
d	m	Diameter	BB	Bubble flow
E	J	Kinetic Energy	D	Dispersed
Eo	-	Eötvös number	DB	Dispersed bubble flow
Fr	-	Froude number	DBF	Dispersed bubble and froth
G	kg/m ² s	Mass flux	DL	Droplet
g	m/s ²	Acceleration due to gravity	EB	Elongated bubble
G/V	-	Ratio of the gravitational force to viscous force	EDB	Elongated bubble and dispersed bubbles
GVF	-	Gas volume fraction	I	Intermittent flow
M	kg/s	Mass flow rate	IW	Inertial wave
N/A	-	Not applicable	PL	Plug flow

NR	-	Not reported	RiW	Ripple wave
Q	m ³ /s	Volumetric flow rate	RW	Roll wave
Re	-	Reynolds number	S	Smooth
Re/Eo	-	Ratio of Reynolds number to Eötvös number	SL	Slug flow
v_m	m/s	Mixture velocity	SLF	Slug and froth
v_{SG}	m/s	Superficial gas velocity	ST	Stratified smooth flow
v_{SG}/v_{SL}	-	Ratio of superficial gas velocity to superficial liquid velocity	SW	Stratified wavy flow
v_{SL}	m/s	Superficial liquid velocity		
We	-	Weber number	Subscript	
We/Eo	-	Ratio of Weber number to Eötvös number	G	Gas phase
x	-	Quality	L	Liquid phase
α	-	Wettability	m	Mixture
λ	-	Baker correction factor	O	Oil phase
φ	-	Baker correction factor	T	Total
Θ	°	Inclination angle		
μ	mPa s	Oil viscosity		
ρ	kg/m ³	Density		
σ	N/m	Surface tension		
ε	-	Void fraction		

508 REFERENCES

- 509 Al-Sheikh, J.N., Saunders, D.E., Brodkey, R.S., 1970. Prediction of flow patterns in
510 horizontal two-phase pipe flow. *Can. J. Chem. Eng.* 48, 21–29.
511 <https://doi.org/10.1002/cjce.5450480105>
- 512 Alves, G.E., 1954. Cocurrent liquid-gas flow in a pipeline contractor. *Chem. Eng. Prog.* 50,
513 449–456.
- 514 Amaya-Gomez, R., López, J., Pineda, H., Urbano-Caguasango, D., Pinilla, J., Ratkovich, N.,
515 Muñoz, F., 2019. Probabilistic approach of a flow pattern map for horizontal, vertical,
516 and inclined pipes. *Oil Gas Sci. Technol.* 74. <https://doi.org/10.2516/ogst/2019034>
- 517 Anumbe, N.C., 2018. Experimental Investigation of Two-Phase (Gas/Liquid) Flow in
518 Intermediate Sized, Horizontal and Inclined Pipes (PhD Thesis). University of South
519 Carolina, Columbia, USA.
- 520 Baba, Archibong-Eso, Aliyu, Ribeiro, Lao, Yeung, 2019. Slug Translational Velocity for
521 Highly Viscous Oil and Gas Flows in Horizontal Pipes. *Fluids* 4, 170.
522 <https://doi.org/10.3390/fluids4030170>
- 523 Bageri, A.S., Gajbhiye, R., Abdulraheem, A., 2017. Evaluating the Effect of Surface Tension
524 on Two-Phase Flow in Horizontal Pipes, in: *SPE Kingdom of Saudi Arabia Annual
525 Technical Symposium and Exhibition*. Society of Petroleum Engineers, Dammam, Saudi
526 Arabia, pp. 2268–2279. <https://doi.org/10.2118/188009-MS>
- 527 Bai, Y., Bai, Q., 2019. Hydraulics, in: *Subsea Engineering Handbook*. Elsevier, pp. 315–361.
528 <https://doi.org/10.1016/B978-0-12-812622-6.00013-0>
- 529 Baker, O., 1954. Simultaneous flow of oil and gas. *Oil Gas J.* 53, 183–195.
- 530 Barnea, D., Shoham, O., Taitel, Y., Dukler, a. E., 1980. Flow pattern transition for gas-liquid
531 flow in horizontal and inclined pipes. Comparison of experimental data with theory. *Int.*

- 532 J. Multiph. Flow 6, 217–225. [https://doi.org/10.1016/0301-9322\(80\)90012-9](https://doi.org/10.1016/0301-9322(80)90012-9)
- 533 Bergelin, O., Gazley, C.J., 1949. Co-Current Gas-Liquid Flow in Horizontal Tubes. Proc.
534 Heat Transf. Fluid Mech. Inst. 29, 5–18.
- 535 Brennen, C.E., 2005. Fundamentals of Multiphase Flow. Cambridge University Press,
536 Cambridge, UK.
- 537 Chen, J.J.J., 1979. Two-phase gas-liquid flow with particular emphasis on holdup
538 measurements and predictions (PhD Thesis). University of Auckland, Auckland, New
539 Zealand.
- 540 Cheng, L., Ribatski, G., Thome, J.R., 2008. Two-phase flow patterns and flow-pattern maps:
541 Fundamentals and applications. Appl. Mech. Rev. 61, 0508021–05080228.
542 <https://doi.org/10.1115/1.2955990>
- 543 Clerc, S., 2000. Numerical Simulation of the Homogeneous Equilibrium Model for Two-
544 Phase Flows. J. Comput. Phys. 161, 354–375. <https://doi.org/10.1006/jcph.2000.6515>
- 545 Coleman, J.W., Garimella, S., 1999. Characterization of two-phase flow patterns in small
546 diameter round and rectangular tubes. Int. J. Heat Mass Transf. 42, 2869–2881.
547 [https://doi.org/10.1016/S0017-9310\(98\)00362-7](https://doi.org/10.1016/S0017-9310(98)00362-7)
- 548 De Lorenzo, M., Lafon, P., Di Matteo, M., Pelanti, M., Seynhaeve, J.M., Bartosiewicz, Y.,
549 2017. Homogeneous two-phase flow models and accurate steam-water table look-up
550 method for fast transient simulations. Int. J. Multiph. Flow 95, 199–219.
551 <https://doi.org/10.1016/j.ijmultiphaseflow.2017.06.001>
- 552 Eaton, B.A., Knowles, C.R., Silberbrg, I.H., 1967. The Prediction of Flow Patterns, Liquid
553 Holdup and Pressure Losses Occurring During Continuous Two-Phase Flow In
554 Horizontal Pipelines. J. Pet. Technol. 19, 815–828. <https://doi.org/10.2118/1525-pa>
- 555 Elliott, A.J., Falcone, G., Putten, D. Van, Leonard, T., Haukalid, K., Pinguet, B., 2021.
556 Investigating reproducibility in multiphase flow metrology : Results from an
557 intercomparison of laboratories. Flow Meas. Instrum. 79, 101951.
558 <https://doi.org/10.1016/j.flowmeasinst.2021.101951>
- 559 Emamzadeh, M., Issa, R.I., 2013. One-dimensional model for numerical simulation of annular
560 flow in horizontal and vertical pipes. Multiph. Sci. Technol. 25, 25–56.
561 <https://doi.org/10.1615/MultScienTechn.v25.i1.20>
- 562 Farokhpoor, R., Liu, L., Langsholt, M., Hald, K., Amundsen, J., Lawrence, C., 2020.
563 Dimensional analysis and scaling in two-phase gas–liquid stratified pipe flow–
564 Methodology evaluation. Int. J. Multiph. Flow 122, 103139.
565 <https://doi.org/10.1016/j.ijmultiphaseflow.2019.103139>
- 566 França, F., Lahey, R.T., 1992. The use of drift-flux techniques for the analysis of horizontal
567 two-phase flows. Int. J. Multiph. Flow 18, 787–801. [https://doi.org/10.1016/0301-9322\(92\)90059-P](https://doi.org/10.1016/0301-9322(92)90059-P)
- 569 Fukano, T., Kariyasaki, A., 1993. Characteristics of gas-liquid two-phase flow in a capillary
570 tube. Nucl. Eng. Des. 141, 59–68. [https://doi.org/10.1016/0029-5493\(93\)90092-N](https://doi.org/10.1016/0029-5493(93)90092-N)
- 571 Govier, G.W., Omer, M.M., 1962. The horizontal pipeline flow of air-water mixtures. Can. J.
572 Chem. Eng. 40, 93–104. <https://doi.org/10.1002/cjce.5450400303>

- 573 Gregory, G.A., Nicholson, M.K., Aziz, K., 1978. Correlation of the liquid volume fraction in
574 the slug for horizontal gas-liquid slug flow. *Int. J. Multiph. Flow* 4, 33–39.
575 [https://doi.org/10.1016/0301-9322\(78\)90023-X](https://doi.org/10.1016/0301-9322(78)90023-X)
- 576 Hoogendoorn, C.J., Buitelaar, A.A., 1961. The effect of gas density and gradual vaporization
577 on gas-liquid flow in horizontal pipes. *Chem. Eng. Sci.* 16, 208–221.
578 [https://doi.org/10.1016/0009-2509\(61\)80032-8](https://doi.org/10.1016/0009-2509(61)80032-8)
- 579 Hoogendoorn, C.J., 1959. Gas-liquid flow in horizontal pipes. *Chem. Eng. Sci.* 9, 205–217.
580 [https://doi.org/10.1016/0009-2509\(59\)85003-X](https://doi.org/10.1016/0009-2509(59)85003-X)
- 581 Humami, F., Dinaryanto, O., Hudaya, A.Z., Widyatama, A., Indarto, Deendarlianto, 2018.
582 Experimental study on the characteristics of flow pattern transitions of air-water two-
583 phase flow in a horizontal pipe, in: *AIP Conference Proceedings*.
584 <https://doi.org/10.1063/1.5049977>
- 585 Jayawardena, S.S., Balakotaiah, V., Witte, L.C., 1997. Flow pattern transition maps for
586 microgravity two-phase flows. *AIChE J.* 43, 1637–1640.
587 <https://doi.org/10.1002/aic.690430627>
- 588 Johnson, H.A., Abou-Sabe, A.H., 1952. Heat transfer and pressure drop for turbulent flow of
589 air-water mixture in a horizontal pipe. *Trans. ASME* 74, 977–987.
- 590 Kaji, R., Azzopardi, B.J., 2010. The effect of pipe diameter on the structure of gas/liquid flow
591 in vertical pipes. *Int. J. Multiph. Flow* 36, 303–313.
592 <https://doi.org/10.1016/j.ijmultiphaseflow.2009.11.010>
- 593 Kokal, S.L., 1987. An experimental study of two-phase flow in inclined pipes (PhD Thesis).
594 University of Calgary. <https://doi.org/10.11575/PRISM/14066>
- 595 Kokal, S.L., Stanislav, J.F., 1989. An experimental study of two-phase flow in slightly
596 inclined pipes-I. Flow patterns. *Chem. Eng. Sci.* 44, 665–679.
597 [https://doi.org/10.1016/0009-2509\(89\)85042-0](https://doi.org/10.1016/0009-2509(89)85042-0)
- 598 Kong, R., Kim, S., 2017. Characterization of horizontal air-water two-phase flow. *Nucl. Eng.*
599 *Des.* 312, 266–276. <https://doi.org/10.1016/j.nucengdes.2016.06.016>
- 600 Kosterin, S., 1949. An investigation of the influence of the diameter and inclination of a tube
601 on the hydraulic resistance and flow structure of gas-liquid mixtures. *Izvestiya Akad.*
602 *Nauk USSR Otd. Tekhnicheskikh Nauk* 12, 1824–1830.
- 603 Lin, P.Y., Hanratty, T.J., 1987. Effect of pipe diameter on flow patterns for air-water flow in
604 horizontal pipes. *Int. J. Multiph. Flow* 13, 549–563.
- 605 Lips, S., Meyer, J.P., 2011. Two-phase flow in inclined tubes with specific reference to
606 condensation: A review. *Int. J. Multiph. Flow* 37, 845–859.
607 <https://doi.org/10.1016/j.ijmultiphaseflow.2011.04.005>
- 608 Mandhane, J.M., Gregory, G.A., Aziz, K., 1974. A flow pattern map for gas-liquid flow in
609 horizontal pipes. *Int. J. Multiph. Flow* 1, 537–553. [https://doi.org/10.1016/0301-](https://doi.org/10.1016/0301-9322(74)90006-8)
610 [9322\(74\)90006-8](https://doi.org/10.1016/0301-9322(74)90006-8)
- 611 Mathure, N., 2010. Study of Flow Patterns and Void Fraction in Horizontal Two-Phase Flow
612 (MSc Thesis). Oklahoma State University, Stillwater, Oklahoma, USA.
- 613 Matsubara, H., Naito, K., 2011. Effect of liquid viscosity on flow patterns of gas-liquid two-

- 614 phase flow in a horizontal pipe. *Int. J. Multiph. Flow* 37, 1277–1281.
615 <https://doi.org/10.1016/j.ijmultiphaseflow.2011.08.001>
- 616 Nguyen, V.T., 1975. Two-phase, gas-liquid concurrent flow: an investigation of holdup,
617 pressure drop and flow pattern in a pipe at various inclinations (PhD Thesis). The
618 University of Auckland, Auckland, New Zealand.
- 619 Ohnuki, A., Akimoto, H., 2000. Experimental study on transition of flow pattern and phase
620 distribution in upward air-water two-phase flow along a large vertical pipe. *Int. J.*
621 *Multiph. Flow* 26, 367–386. [https://doi.org/10.1016/S0301-9322\(99\)00024-5](https://doi.org/10.1016/S0301-9322(99)00024-5)
- 622 Osundare, O.S., Falcone, G., Lao, L., Elliott, A., 2020. Liquid-Liquid Flow Pattern Prediction
623 Using Relevant Dimensionless Parameter Groups. *Energies* 13, 4355.
624 <https://doi.org/10.3390/en13174355>
- 625 Saha, S.K., 2015. *Microchannel Phase Change Transport Phenomena*. Butterworth-
626 Heinemann publications.
- 627 Shao, N., Gavriilidis, A., Angeli, P., 2009. Flow regimes for adiabatic gas-liquid flow in
628 microchannels. *Chem. Eng. Sci.* 64, 2749–2761.
629 <https://doi.org/10.1016/j.ces.2009.01.067>
- 630 Sharma, H., Das, G., Samanta, A.N., 2006. ANN-based prediction of two-phase gas- liquid
631 flow patterns in a circular conduit. *AIChE J.* 52, 3018–3028.
632 <https://doi.org/10.1002/aic.10922>
- 633 Shi, J., Lao, L., Yeung, H., 2017. Water-lubricated transport of high-viscosity oil in
634 horizontal pipes: The water holdup and pressure gradient. *Int. J. Multiph. Flow* 96, 70–
635 85. <https://doi.org/10.1016/j.ijmultiphaseflow.2017.07.005>
- 636 Shi, J., Yeung, H., 2017. Characterization of liquid-liquid flows in horizontal pipes. *AIChE J.*
637 63, 1132–1143. <https://doi.org/10.1002/aic.15452>
- 638 Shoham, O., 2005. *Mechanistic Modeling of gas liquid two phase flow in pipes*. The Society
639 of Petroleum Engineers (SPE), Tulsa.
- 640 Shoham, O., 1982. *Flow Pattern Transition and Characterization in Gas-Liquid Two-Phase*
641 *Flow in Inclined Pipes* (PhD Thesis). Tel-Aviv University, Tel Aviv, Isreal.
- 642 Simpson, H.C., Rooney, D.H., Grattan, E., Al-Samarrae, F.A.A., 1977. Two-phase flow
643 studies in large diameter horizontal tubes., in: *European Two-Phase Flow Group*
644 *Meeting*.
- 645 Spedding, P.L., Chen, J.J.J., 1981. A simplified method of determining flow pattern transition
646 of two-phase flow in a horizontal pipe. *Int. J. Multiph. Flow* 7, 729–731.
647 [https://doi.org/10.1016/0301-9322\(81\)90042-2](https://doi.org/10.1016/0301-9322(81)90042-2)
- 648 Spedding, P.L., Chen, J.J.J., 1980. Two Phase Flow Pressure Loss Prediction, in: *National*
649 *Conference Publication - Institution of Engineers, Australia*. Brisbane, pp. 420–423.
- 650 Spedding, P.L., Hand, N.P., Spence, D.R., 1989. Data on horizontal, co-current, two phase
651 gas-liquid flows.
- 652 Spedding, P.L., Nguyen, V.T., 1980. Regime maps for air water two phase flow. *Chem. Eng.*
653 *Sci.* 35, 779–793. [https://doi.org/10.1016/0009-2509\(80\)85062-7](https://doi.org/10.1016/0009-2509(80)85062-7)
- 654 Todkar, S.R., Kapale, U.C., Anil, T.R., Chapgaon, A.N., 2016. *An Experimental Study on*

655 Two Phase (Air-Water) Flow Characteristics in a Horizontal Pipe at Atmospheric
656 Conditions. *Int. J. Eng. Res. Technol.* 5, 687–692.

657 Troniewski, L., Ulbrich, R., 1984. The analysis of flow regime maps of two-phase gas-liquid
658 flow in pipes. *Chem. Eng. Sci.* 39, 1213–1224. [https://doi.org/10.1016/0009-](https://doi.org/10.1016/0009-2509(84)85082-4)
659 [2509\(84\)85082-4](https://doi.org/10.1016/0009-2509(84)85082-4)

660 Tzotzi, C., Bontozoglou, V., Andritsos, N., Vlachogiannis, M., 2011. Effect of fluid
661 properties on flow patterns in two-phase gas-liquid flow in horizontal and downward
662 pipes. *Ind. Eng. Chem. Res.* 50, 645–655. <https://doi.org/10.1021/ie100239v>

663 Vásquez, F., Stanko, M., Vásquez, A., De Andrade, J., Asuaje, M., 2012. Air-water: Two
664 phase flow behavior in a horizontal pipe using computational fluids dynamics (CFD).
665 *WIT Trans. Eng. Sci.* 74, 381–392. <https://doi.org/10.2495/AFM120341>

666 Weisman, J., Duncan, D., Gibson, J., Crawford, T., 1979. Effects of Fluid Properties and Pipe
667 Diameter on Two- Phase Flow Patterns in Horizontal Lines. *Int. J. Multiph. Flow* 5,
668 437–462. [https://doi.org/10.1016/0301-9322\(79\)90031-4](https://doi.org/10.1016/0301-9322(79)90031-4)

669 White, P.D., Huntington, R.L., 1955. Horizontal Co-Current Two-Phase Flow of Fluids in
670 Pipe Lines. *Pet. Eng.* 27, D40–D45.

671 Wong, T.N., Yau, Y.K., 1997. Flow patterns in two-phase air-water flow. *Int. Commun. Heat*
672 *Mass Transf.* 24. [https://doi.org/https://doi.org/10.1016/S0735-1933\(96\)00110-8](https://doi.org/https://doi.org/10.1016/S0735-1933(96)00110-8)

673 Xiao, J.J., Shoham, O., Brill, J.P., 1990. Comprehensive mechanistic model for two-phase
674 flow in pipelines. *Proc. - SPE Annu. Tech. Conf. Exhib. Pi*, 167–180.

675 Yang, C.-Y., Shieh, C.-C., 2001. Flow pattern of air–water and two-phase R-134a in small
676 circular tubes. *Int. J. Multiph. Flow* 27, 1163–1177. [https://doi.org/10.1016/S0301-](https://doi.org/10.1016/S0301-9322(00)00070-7)
677 [9322\(00\)00070-7](https://doi.org/10.1016/S0301-9322(00)00070-7)

678 Zhang, H.Q., Wang, Q., Sarica, C., Brill, J.P., 2004. Unified model for gas-liquid pipe flow
679 via slug dynamics - Part 1: Model development. *SPE Repr. Ser.* 125, 44–51.
680 <https://doi.org/10.1115/1.1615246>

681 Zhao, L., Rezkallah, K.S., 1993. Gas-liquid flow patterns at microgravity conditions. *Int. J.*
682 *Multiph. Flow* 19, 751–763. [https://doi.org/10.1016/0301-9322\(93\)90041-R](https://doi.org/10.1016/0301-9322(93)90041-R)

683

684 5 APPENDIX A

685 5.1 Development of flow regime maps for gas-liquid flow in horizontal pipelines

686 Different research groups have employed various combinations of coordinates for presenting
687 flow regime maps in a 2D graph. Table 7 shows the chronological developments of flow regime
688 maps over the years for gas-liquid flow in horizontal pipelines.

689 Table 7: Development of flow regime maps for gas-liquid in a horizontal pipeline.

Author	Pipe size (mm)	Fluids	Coordinates	Identified flow patterns
Bergelin and Gazley (1949)	25.4	Air-water	Gas mass flow rate (M_G) versus liquid mass flow rate (M_L)	NR
Kosterin (1949)	25.4, 50.8,	Air-water	Mixture velocity (v_m) versus input gas volume fraction (v_{SG}/v_m)	NR

	76.2, 101.6			
Johnson and Abou-Sabe (1952)	22.1	Air-water	Gas mass flow rate (M_G) versus liquid mass flow rate (M_L)	NR
Alves (1954)	26.6	Air-water; Air-oil	Superficial gas velocity (v_{SG}) versus superficial liquid velocity (v_{SL})	BB, SL, PL, ST, SW, AN, spray
Baker (1954)	25.4- 101.6	Air-water; NG-oil	G_G/λ and $G_L\lambda\phi/G_G$ (λ , ϕ are parameters which account for different gas and liquid properties)	ST, SW, PL, SL, AN, BB or froth and D
White and Huntington (1955)	25.4; 38.1; 50.8.	Air-water; Air-oil; NG-oil	Liquid mass flux (G_L) versus gas mass flux (G_G)	NR
Hoogendorn (1959)	24-140 ^S ; 50 ^r	Air-water	Mixture velocity (v_m) versus input gas volume fraction (v_{SG}/v_m)	ST, SW, PL, SL, mist-AN, and froth
Govier and Omer (1962)	26.1	Air-water	Liquid mass flux (G_L) versus gas mass flux (G_G)	ST, SW, PL, SL, BB, AN
Eaton et al. (1967)	50.8; 101.6.	NG-water; NG-crude oil; NG-distillate	Two-phase Reynolds number (Re_{tp}) versus two-phase Weber number (We_{tp})	NR
Al-Sheikh et al. (1970)	18.7- 154.1	Gas-liquid	10 coordinates derived from the authors 9 correlations.	BB, PL, ST, SW, SL, AN, spray
Mandhane et al. (1974)	12.7- 165.1	Gas-liquid	Superficial liquid velocity (v_{SL}) versus superficial gas velocity (v_{SG})	BB, ST, SW, SL, AN-mist, DB
Simpson et al. (1977)	127; 216	Air-water	Superficial liquid velocity (v_{SL}) versus superficial gas velocity (v_{SG})	N/A
Weisman et al. (1979)	12; 25; 51	Air-water; Air-glycerol	Gas mass flux (G_G) versus liquid mass flux (G_L)	PL, BB, ST, SW, SL, AN, D
Spedding and Nguyen (1980)	45.5	Air-water	Froude number ($v_m/\sqrt{g d}$) versus volumetric flow ratio (Q_L/Q_G)	13 distinguishable flow patterns lumped to 4 types of flow: stratified, bubble & slug, droplets, mixed
Spedding and Chen (1981)	13-50	Gas-liquid	Superficial liquid velocity (v_{SL}) versus superficial gas velocity (v_{SG})	AN, PL or long BB, ST+ripple, SL, ST, ST+roll wave
Lin and Hanratty (1987)	25.4; 95.3	Air-water	Superficial liquid velocity (v_{SL}) versus superficial gas velocity (v_{SG})	ST, SW, SL, AN, pseudo-SL
Kokal and Stanislav (1989); Kokal (1987)	25.8; 51.2; 713	Air-light oil	Superficial liquid velocity (v_{SL}) versus superficial gas velocity (v_{SG})	ST, SW, AW, EB, EDB, SL, SLF, DB, DBF
Xiao et al. (1990)	50	Air-water	Superficial liquid velocity (v_{SL}) versus superficial gas velocity (v_{SG})	ST, SW, AN, DB, I
Wong and Yau (1997)	12	Air-water	Superficial liquid velocity (v_{SL}) versus superficial gas velocity (v_{SG})	16 distinct flow patterns lumped to 4 flow types: i. ST: S, IW, RiW, RW

				ii. I: PL, SL, PL-SL, pseudo-SL iii. DB iv. MF: RW+DL, RW+DL+AN, RW+AN, pseudo-SL+AN, pseudo-SL+thin AN, SL-AN, DB+SL
Jayawardena et al. (1997)	9.5-40	Gas-liquid		BB, SL, AN
Tzotzi et al. (2011)	24	He-water; Air-water; CO ₂ -water; Air-aqueous butanol	-Superficial liquid velocity (v_{SL}) versus superficial gas velocity (v_{SG}) - Superficial liquid velocity (v_{SL}) versus superficial gas velocity multiply by the square root of ratio of liquid density to gas density ($v_{SG} * (\rho_L/\rho_G)$)	ST, SL, SL-froth, AN
Kong and Kim (2017)	38.1	Air-water	Superficial liquid velocity (v_{SL}) versus superficial gas velocity (v_{SG})	BB, PL, SL, ST, SW and AN

690 Note: others means used data from others studies; ^s and ^r indicate smooth and rough pipe, respectively; NG means Natural-
691 gas, NR means not reported; ST: stratified; SW: stratified wavy; AN: annular; EB: elongated bubble; EDB: elongated bubble
692 and dispersed bubbles; SL: slug; SLF: slug and froth; DB: dispersed bubble; DBF: dispersed bubble and froth; BB: bubble; D:
693 dispersed; PL: plug; I: intermittent, IW: inertial wave; RiW: ripple wave; RW: roll wave; DL: droplet; S: smooth.

694 Based on Table 7, the superficial liquid velocity (v_{SL}) versus superficial gas velocity (v_{SG}) is
695 the most frequently used for constructing flow maps in recent times. This may be due to the
696 assumption that transition curves are least sensitive to changes in pipe diameter and fluid
697 properties when superficial liquid velocity (v_{SL}) and superficial gas velocity (v_{SG}) are used as
698 coordinates (Lin and Hanratty, 1987).

699 The choice of coordinate systems (ordinate and abscissa) used for constructing a flow pattern
700 map is divided into three groups, according to Cheng et al. (2008); Troniewski and Ulbrich
701 (1984):

- 702 1. Phase velocities or fluxes (group I): The parameters include gas and liquid superficial
703 velocities (v_{SG} , v_{SL}), gas and liquid mass flow rates (M_G , M_L), gas and liquid mass fluxes
704 (G_G , G_L). They are convenient to employ but cannot create universal flow pattern maps for
705 different two-phase combinations.
- 706 2. Quantities that refer to the homogeneous model of a two-phase flow (group II): They are
707 the transformation of parameters from group I, which include total velocity (v_T), total mass
708 flux (G_T), Froude number based on total velocity (Fr_T), void fraction (ϵ), and quality (x).
709 Valid only for some flow patterns.
- 710 3. Parameters comprising the physical properties of the phases (group III): These include gas
711 and liquid Reynolds numbers (Re_G , Re_L), Baker correction factors (λ , ϕ), gas and liquid
712 kinetic energies (E_G , E_L). This group has the best potential to provide universal flow
713 regime maps.

714