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5 I. NOMENCLATURE

D_{eq}	Equivalent duct diameter [m]
H	Test section height [m]
L	Test section length [m]
M	Mach number
P	Pressure [Pa]
T	Temperature [K]
t	Time [s]
U	Component i of the velocity vector [m/s]
W	Test section width [m]
x	Streamwise component of the position vector [m]
δ	Boundary layer thickness [mm]

Subscript

b	Back-pressure
0	Total condition

6 II. INTRODUCTION

7 During the flight of a ramjet or a scramjet, the low-density air enters via the engine
8 inlet, where it is compressed through an extremely complex mechanism before reaching the
9 combustor. Between the inlet and the combustor, a nearly parallel duct, called an isolator,
10 is placed to prevent the interaction of the flow at the inlet with that inside the combustion
11 chamber.¹ The combustion of fuel causes a rapid pressure rise in the combustion chamber
12 and the formation of a shock structure inside the isolator results in different conditions
13 upstream and downstream of the flow passage. This flow structure, composed of a series
14 of shock waves, is called a shock train. The ability to accurately predict and control such
15 a shock wave structure would provide a means to enhance the performance of flow devices
16 operating at high speeds such as ramjets and scramjets, the engine efficiency, or the mixing
17 of fuel injected from the combustor walls.² Other relevant applications characterised by the

18 presence of shock trains include supersonic compressors, ejectors, and wind-tunnel diffusers.³

19 The shock train system has demonstrated to be largely dependant on the geometry and
20 the flow conditions at the two extremities of the duct.^{4,5} In particular, the ratio of bound-
21 ary layer thickness to duct equivalent hydraulic diameter, δ/D_{eq} , also referred to as flow
22 confinement, is one of the leading variables that determines the configuration of the shock
23 train.⁶⁻⁹ Morgan et al.¹⁰ found that the local flow blockage is more important than the total
24 pressure loss in locating the initial shock within an isolator. Weiss et al.¹¹ confirmed that
25 the confinement level and Mach number are the dominant variables which characterise the
26 position and length of the shock train, whereas the Reynolds number has a much smaller
27 effect.

28 Figure 1 schematically illustrates the coupling between the shock train and the boundary
29 layer for inflow Mach numbers greater than 1.5. The flow enters the inlet at supersonic
30 speeds and is decelerated to subsonic velocity behind the first normal shock wave, *NSW*, in
31 the core flow. The pressure rise is transmitted upstream through the boundary layer region,
32 causing a thickening of the boundary layer itself. The growth of the boundary layer deflects
33 the streamline forming an oblique shock, *FOS*. Since the flow remains supersonic behind
34 the front oblique shock, a rear oblique shock wave, *ROS*, forms behind it. The two oblique
35 shocks converge into the triple point, *TP*, and combine with the initial normal shock into a
36 λ shock structure, λS . At the point of bifurcation, a shear layer, *SL*, develops, as it can be
37 observed in the form of slip lines. In the region confined between the slip lines, the stronger
38 deceleration through the normal shock produces a misalignment of the flow velocity with
39 the outer parts where the flow passes through the two oblique shocks. The thickening of the
40 boundary layer reduces the effective area of the core flow, so that the subsonic flow behind
41 the rear oblique shock wave, *ROS*, is accelerated again to supersonic velocity. At this point
42 the supersonic flow interacts with the thick boundary layer and the same process is repeated
43 several times up to a terminal shock after which the flow is subsonic in the entire cross
44 section.

45 The numerous variables which affect the shock train configuration make a comprehen-
46 sive analysis of the flow field extremely difficult. Some flow measurements in shock trains
47 cannot be experimentally obtained and key mechanisms, such as the interaction between
48 three-dimensional shock waves and recirculation zones, are too complex to be analysed and
49 explained by experiments alone.^{12,13} These limitations have led industry towards an increas-

50 ing use of computational analysis to estimate the flow physics and to design flow devices
51 with adequate performance.^{14,15}

52 The intrinsic problem of numerical methods is that the domain of interest must be di-
53 vided into cells where a chosen numerical method is applied to solve differential equations,
54 introducing an approximation that differs from the exact solution. The Navier-Stokes (NS)
55 equations are widely employed because they allow the simultaneous solving of the viscous
56 and inviscid flow fields. However, computations which include the interaction between shock
57 waves and turbulence are highly sensitive to the turbulence closure model.^{8,16} The shear
58 stress transport model (SST) was successfully used by Saha et al.¹⁷ to predict the wall pres-
59 sure in an intake with freestream Mach number from 3 to 8, but the numerical simulation
60 performed by Gawehn¹⁸ strongly deviated from the experimental findings. By using the
61 Reynolds stress transport models (RSM) Mousavi et al.¹⁹ successfully predicted the posi-
62 tion and shape of the shock train in a convergent-divergent nozzle. Sun et al.²⁰ obtained
63 good agreement with the experimental data using the algebraic Baldwin-Lomax turbulence
64 model with only one value of the tested back-pressures. The reason of the limited accuracy
65 of the algebraic turbulence model is the Boussinesq approximation, which prevents its use
66 in separated flows.²¹⁻²⁴ Although Chan et al.¹⁴ demonstrated that the k - ω Wilcox model
67 is suitable for supersonic and hypersonic aerothermodynamic applications, at the NASA
68 Langley Research Center, all the models used by Baurle et al.²⁵ failed to accurately predict
69 the shape and extent of the separated flow region caused by the shock wave/boundary layer
70 interactions in a scramjet isolator.

71 Most studies have concentrated on two-dimensional simulations^{13,26} even though an ade-
72 quate description of a three-dimensional flow with a two-dimensional model is unreasonable.²⁷
73 Three-dimensional investigations provide more accurate insight into the effect of the four
74 walls surrounding rectangular ducts on the complex characteristics of the shock train.²⁸
75 To reduce computational time, Sridhar et al.²⁹ used only one-quarter of the actual duct
76 as computational domain consequently, the results displayed a symmetrical flow field, in
77 contrast with the experimental findings.

78 Additionally, research on high-speed isolators has mainly focused on cylindrical ducts, and
79 only recently on rectangular cross-sections. This choice is due to the fact that the axisym-
80 metric configuration minimises the three-dimensional effects from the shock wave/boundary
81 layer interactions encountered in rectangular channels.³⁰ The numerical and experimental

82 results by Kawatsu et al.³¹ reported that in rectangular ducts, the boundary layer separation
 83 occurs only near the corners of the duct but not at the centre plane of the test section, as it
 84 is observed with schlieren photography. Although Billig et al.³² stated that since the trend of
 85 the pressure rise for cylindrical and rectangular cross-sections is quite similar then the shock
 86 train characteristics may also be similar, no similarity law linking different cross-sectional
 87 geometries have been reported. In contrast, differences have been highlighted by Lin et al.³³
 88 observing that, compared to rounded cross-sectional area ducts, in the rectangular configu-
 89 ration the pressure profile of the shock train initially rises steeply, reaches a maximum value
 90 early, and drops quickly at the isolator exit. Also, the maximum pressure rise is smaller,
 91 independent of the Mach number. These differences were attributed to the fact that in the
 92 rectangular duct, the larger cross-sectional perimeter and the presence of the four corners
 93 lead to an increased cross-sectional area of the duct covered by the boundary layer, thus
 94 reducing the effective free-stream area. On the other hand, for the same Mach number, the
 95 leading edge of the shock train was detected to be roughly at the same axial position inside
 96 the isolator for both circular and rectangular cross-sections.

97 The present study analyses the sensitivity to the variables that influence the character-
 98 istics of the shock system which establish in a long duct. The effects of the choice of the
 99 turbulence model and the use of a three-dimensional domain are investigated.

100 III. NUMERICAL AND PHYSICAL SETUP

101 To validate the numerical approach, the Mach 2 shock train experimentally studied by
 102 Sun et al.^{13,20} in a square duct was replicated. The boundary and geometrical conditions are
 103 reported in Table I. The cross section and length of the test section are $80 \times 80 \text{ mm}^2$ and 1500

M	$T_0[K]$	$P_0[kPa]$	$P_b[kPa]$	$H[mm]$	$W[mm]$	$L[mm]$	δ/D_{eq}
2	300	196	92.2	80	80	880	0.25

104 Table I. Boundary and geometry conditions of the computational domain of the validation model.²⁰ The
 105 subscript 0 refers to the total condition and P_b is the back-pressure.

106 mm , with a length to equivalent diameter ratio L/D_{eq} of 18.75. Along with experiments,
 107 Sun et al.^{13,20} performed a numerical investigation with a computational domain length of
 108 11 times the height starting from $L/D_{eq} = 7$. Since the effect of the flow confinement, δ/D_{eq} ,

109 at the inlet of the computational domain plays a fundamental role in the location of the
110 shock train, to match the experimental conditions of $\delta/D_{eq}= 0.25$, Sun et al.²⁰ imposed a
111 velocity profile given by the 1/7-power law at the inlet.

112 The numerical simulations were carried out by solving the two-dimensional coupled im-
113 plicit Reynolds-averaged Navier-Stokes (RANS) equations in STAR-CCM+³⁴. In real su-
114 peronic air-breathing engines, the shock train is inherently unsteady due to the combustion
115 instabilities.³⁵ However, longitudinal fluctuations around the averaged position are small
116 and can be assessed in a steady manner. The $k-\omega$ Wilcox turbulence model was used in
117 most cases. This model is able to reproduce subtle features close to the solid boundary
118 and is more accurate for two-dimensional boundary layers with both favourable and adverse
119 pressure gradients, and in the presence of separation induced by the interaction with a shock
120 wave.³⁶

121 The RANS equations are discretised using the cell-centred finite volume method. The
122 inviscid and viscous fluxes are evaluated using respectively the Liou's AUSM+ flux-vector
123 splitting scheme based on the upwind concept and the second-order central differences.

124 The working fluid is approximated as an ideal gas. The viscosity and thermal conduc-
125 tivity are evaluated using Sutherland's law. Adiabatic and no-slip boundary conditions are
126 imposed on the walls along the duct. Initial conditions are set with an inviscid normal shock
127 at the exit of the computational domain. At the outlet boundary the flow variables except
128 pressure are extrapolated from the adjacent cell value using reconstruction gradients. The
129 back-pressure was determined from the experimental results to be approximately $P_b= 92.2$
130 kPa and assumed constant at the exit plane.

131 The static pressure and Mach number distributions along the duct obtained by Sun et
132 al.¹³ through numerical simulations and experiments are shown in Figure 2. Two values of
133 the back-pressure, $P_b= 92.2 kPa$ (case A) and $96.6 kPa$ (case B), are compared for an inlet
134 Mach number of 2. It can be observed that the experimental pressure data at the wall of
135 case B are well replicated with the numerical simulations. For case A, although the location
136 of the first shock wave matches the experimental findings, the pressure distribution is not
137 well-resolved. The poor accuracy of the numerical results obtained by Sun et al.^{13,20} due to
138 the use of an inadequate turbulence model to describe separated flows is an important aspect
139 to take into account when the discrepancies with the current numerical code are analysed.
140 Additionally, the only experimental data available are from the pressure tapping at the wall,

141 whereas pressure and Mach number distributions at the duct centreline are obtained with
142 computation only. Therefore, only the wall pressure distributions are considered reliable to
143 make comparisons.

144 The computational domain used in the current study is formed of a rectangular block.
145 Due to the symmetry of the problem, half of the region of the flow field is computed in
146 the two-dimensional case, and one quarter in the three-dimensional case. The mesh is
147 composed of structured quadrilateral cells and the grid points are clustered towards the wall
148 to resolve the behaviour of the boundary layer. Refinements are necessary in the regions
149 where the gradients are known to be relevant and the thickness of the closest cell to the wall
150 is important for the accuracy of the results. Figure 3 shows the structure of the numerical
151 grid employed, where $y/D_{eq} = 0$ corresponds to the wall and $y/D_{eq} = 0.5$ is the centreline of
152 the duct.

153 IV. RESULTS AND DISCUSSION

154 A. Two-dimensional Grid Convergence

155 In a narrow channel, typical of this kind of flows, the ratio of the flow confinement ahead
156 of the shock train to the duct height plays a fundamental role on the location and length
157 of the shock train. Without a boundary layer at the inlet of the computational domain the
158 shock train would begin further downstream in the duct, in agreement with Huang et al.³⁷
159 However, the viscous effects near the wall reduce the flow speed and also the effective area of
160 the duct. This leads to a high sensitivity of the shock train to the length of the computational
161 domain. In the results achieved by Sun et al.¹³ a portion of duct with length $L/D_{eq} = 11$ was
162 taken to process the data, with the inlet located at δ/D_{eq} equal to approximately 0.25. To
163 replicate the same inlet conditions, an iterative process of mesh refinement and duct length
164 analysis was performed. An initial simulation was run to extract the flow properties at a
165 specific axial location, which are then imposed at the inlet of another simulation as a fixed
166 boundary condition. Figure 4 illustrates that by imposing a boundary layer profile at the
167 inlet of the computational domain the shock train establishes in the same manner as with
168 the case in which the boundary layer naturally develops along the duct walls.

169 The imposition of the boundary layer at the inlet of the computational domain requires

170 a numerically expensive procedure since two simulations need to be run. Therefore, in
 171 the current study the boundary layer was left to develop along the duct wall as it occurs
 172 naturally in the experiments by using a computational domain of length $L/D_{eq} = 23$. This
 173 value ensures an inlet Mach number equal to 2 ahead of the shock train and establishes the
 174 boundary layer similar to the reference study.¹³ Only the portion of the duct with a length
 175 11 times the height was taken to process the data, with the inlet located at δ/D_{eq} equal to
 176 approximately 0.25.²⁰

177 Since the quality of the numerical solution mainly depends on the size of the grid cells
 178 and their distribution in the computational domain, seven grids, tabulated in Table II, are
 179 employed to find the optimal combination between the requirements of adequate accuracy
 180 and computational resources. Except for Grid 1, for all the finer grids the value of the wall

Grid	1	2	3	4	5	6	7
N_x	368	921	2454	4601	6134	9200	12268
N_y	62	116	154	276	314	350	452

Table II. Number of cells in different grids.

181
 182

183 y^+ is smaller than unity in the entire domain, providing a good resolution of the boundary
 184 layer gradients.

185 The flow properties distributions illustrated in Figure 5 have been shifted by the location
 186 of the initial shock wave so that they start at the same axial coordinate. The wall static
 187 pressure monotonically increases due to the diffusing effect of the boundary layer. On the
 188 other hand, the peaks in the centreline pressure plot identify the individual waves composing
 189 the shock train which are gradually damped along the duct. Although the general behaviour
 190 of the shock train is similar in the seven cases, as the mesh resolution increases, the shock
 191 train moves upstream towards the inlet and increases in length. This is caused by the fact
 192 that a coarse mesh fails to adequately resolve the fine structures such as the boundary layer.
 193 Fine grids better match the experimental data because the representation of the flow field is
 194 more accurate. Since the back-pressure is prescribed as a boundary condition, the pressure
 195 at the end of the shock train always converges towards the experimental value. These results
 196 agree with all cases in literature despite the contradictory finding by Carroll et al.,²⁴ who
 197 observed that a grid refinement in the transverse direction only causes the shock train to
 198 move toward the exit plane.

199 From Figure 5, as the grid is refined, the difference between two subsequent pressure
 200 profiles gradually decreases and the location of the shock train tends to stabilise at a fixed
 201 axial coordinate. The difference between Grid 6 and Grid 7 is not significant and the relative
 202 error is less than 1.2%. The relative error in the axial coordinate of the shock train between
 203 Grid 4 and Grid 7 is approximately 8%. However, Figure 6 show very little changes for grids
 204 finer than Grid 4. The difference in the magnitude of the pressure peaks of the first and
 205 second shocks, respectively peak 1st shock and peak 2nd shock, and the pressure recovery
 206 behind the 1st shock converge towards asymptotic values. The variation in magnitude of
 207 the first and second shock between Grid 4 and Grid 7 is less than 0.2%, as it is also evident
 208 from the centreline pressure profile, in Figure 5(b). Taking into account both the accuracy
 209 of the grid and the computational cost, Grid 6 is used to perform the simulations reported
 210 in the present study apart from the three-dimensional case when, due to the large number
 211 of cells, Grid 4 is used.

212 B. Effect of Turbulence Model

213 The influence of using three different turbulence models, the $k-\omega$ Wilcox, $k-\omega$ Menter
 214 SST, and $k-\varepsilon$ realisable, is investigated. As Figure 7 illustrates, compared to the $k-\omega$ Wilcox
 215 model, the magnitude of the density gradient obtained with the $k-\omega$ Menter SST and $k-\varepsilon$
 216 realisable models show several differences. From the close up in Figures 8(b) and 8(c) the
 217 leading shock wave is not normal at the centre of the duct. The front shock has a χ shape,
 218 identified as χS , although in the $k-\varepsilon$ case, in Figure 8(c), a slip line, SL , at the centreline
 219 is visible. It is interesting to note that in the $k-\omega$ Menter SST case, in Figure 8(b), a weak
 220 slip line is present just behind the first shock wave. A second shock occurs and is linked
 221 to the rear legs of the oblique shocks at the edge of the boundary layer. This latter shock,
 222 SN , is normal in a small portion at the centreline of the duct and decelerates the flow to
 223 subsonic conditions. The centreline pressure distribution, in Figure 9(a), illustrates that
 224 with the $k-\omega$ Menter SST the initial pressure rise is not composed of a single peak. The flow
 225 passing through the χ shock is decelerated but remains in the supersonic range. The flow
 226 is further decelerated to subsonic speeds through the normal shock that corresponds to the
 227 steep pressure rise in the first pressure peak.

228 From the pressure distributions in Figure 9, both the wall and centreline pressure converge

229 to the value of the back-pressure imposed as the boundary condition. The wall pressure in
 230 the $k-\omega$ Menter SST model is underpredicted, as visible in Figure 9(b). Although the $k-\varepsilon$
 231 model locates the shock train several L/D_{eq} downstream in the duct, it predicts the shock
 232 train structure more accurately. Compared against the $k-\omega$ Wilcox, the spacing between
 233 consecutive shocks is quite accurate but the amplitude of the shocks behind the leading
 234 shocks is overpredicted and the latter shocks in the shock train structure are very weak.
 235 The considerably shorter shock train length may also be a contributing factor. The $k-\omega$
 236 Menter SST model predicts a slightly longer shock train but it fails to locate the several
 237 shock waves of the shock train system on the axial coordinates. The absence of a normal
 238 portion of the leading shock at the centreline contributes to the failure in capturing the
 239 subsonic flow following the first shock and consequently the entire structure of the shock
 240 train is affected.

241 The establishment of the shock train in the duct mainly depends on the way the boundary
 242 layer develops on the walls, and hence a model capable to accurately reproduce the subtle
 243 features close to the solid boundary plays a fundamental role. Although the three turbulence
 244 models employed are two-equation models, only the $k-\omega$ Wilcox fulfils this requirement. As
 245 shown in Figure 5, the $k-\omega$ Wilcox closely matches the reference data by Sun et al.¹³ of the
 246 entire shock train in terms of flow properties, location of the shock train, distance between
 247 shocks, and shock strength. There is considerable evidence in the literature that the $k-\omega$
 248 model is more computationally robust than the $k-\varepsilon$ model for the description of turbulent
 249 flows close to a solid boundary.³⁸ The $k-\omega$ Menter SST includes the $k-\varepsilon$ model in the far-
 250 field through a blending function. This demonstrates the inability of the $k-\varepsilon$ to describe the
 251 shock train characteristics even in the core flow. On the other hand, the $k-\omega$ Wilcox model
 252 confirms to be the most suitable for describing the shock train behaviour in internal ducts.

253 C. Sidewalls Effects

254 Two-dimensional simulations have the advantage of being efficient since the inclusion
 255 of the third dimension costs additional computational time. However, the presence of the
 256 sidewalls cannot be neglected when the duct aspect ratio is unity. Grid 4 is used to generate
 257 the two-dimensional computational domain. The same grid structure with the addition of
 258 the third dimension is applied to the three-dimensional domain. Due to the symmetry of

259 the duct, one quarter of the experimental geometry is simulated in the three-dimensional
260 case with a grid composed of 28 million cells.

261 As visible from Figure 10 the location of the shock train in the 3D case occurs with
262 an apparent thin boundary layer. In reality, compared to the 2D case, in 3D the shock
263 train occurs further upstream because of the effect of the boundary layer on the side walls
264 and the corners. In the 2D case, the boundary layer develops only on the top and bottom
265 walls of the test section, but in the 3D case the boundary layer on the side walls and
266 the corners is also a contributing factor. Since the duct is of square cross-sectional area,
267 the boundary layer on the side walls affects the flow in the same extent as the top and
268 bottom walls. Consequently, with the inclusion of the boundary layer from all walls, the
269 flow confinement reaches approximately the same value as in the 2D case. This demonstrates
270 that flow confinement plays the greatest role in determining the location of the initial shock,
271 in agreement with the literature.^{10,33}

272 Figure 11 illustrates the numerical schlieren and Mach number contour on different cross
273 sections. The location x_1 identifies the cross section corresponding to the initial pressure
274 rise and the subsequent planes are spaced by $L/D_{eq} = 1$ apart. The cross sections are better
275 displayed in Figure 12. The results show a large separation region at the corners downstream
276 of the leading shock from x_1 . The separation extends along the entire duct reducing the
277 core flow. The several shock waves forming the shock train gradually decelerate the core
278 flow so that from approximately x_4 the shocks are very weak. From x_5 the flow structure
279 shows little changes and at approximately x_6 the shock train is terminated.

280 Figure 13 compares the Mach number and pressure profiles obtained with the 2D and
281 3D simulations. The plots are shifted for common pressure rise and normalised with the
282 equivalent hydraulic diameter. The pressure profiles, in Figure 13(a), illustrates a small
283 difference between the two cases. The centreline pressure shows that the shape of the shock
284 train is similar in the two cases and, in particular, the first shock wave is captured with
285 the same strength. On the other hand, the flow behind the first shock is decelerated more
286 strongly in 3D, as the deeper trough illustrates. The reason of such a difference is due to the
287 thinner boundary layer behind the leading shock which allows the flow to expand more in the
288 subsonic region. This is believed to cause the non-perfect matching of the subsequent shock
289 waves composing the shock train. As previously explained, the first shock is responsible
290 of determining the shape of the entire shock train structure. The same trend is visible

291 from the Mach number profile in Figure 13(b): since the flow conditions of the incoming
292 flow ahead of the shock train are the same in both cases, the strength of the leading shock
293 matches excellently. However, behind the leading shock the subsequent shocks differ. It
294 emerges that in 3D simulations at the end of the shock train the flow is decelerated to a
295 lower Mach number. The lack of experimental data cannot confirm the real Mach number
296 variation through the shock train. Therefore, taking into account the limitation due to the
297 absence of the sidewall effects, two-dimensional simulations are still useful for the qualitative
298 understanding of the mechanism of formation of the shock train in long ducts.

299 V. CONCLUSIONS

300 The formation of a shock train structure in an air-breathing engine prevents the inter-
301 action of the flow at the inlet with that inside the combustion chamber guaranteeing that
302 the air entering the combustor is decelerated to lower speeds. The understanding of such a
303 flow structure is vital for the improvement of the design of high-speed engines as well as the
304 development of flow control methodologies.

305 This investigation on a shock train at inflow Mach number of 2 in a rectangular duct
306 has demonstrated the high sensitivity of the shock train to the solving equations. Since the
307 shock train establishment in the duct is caused by the interaction with the boundary layer,
308 the flow confinement has demonstrated to be the key parameter in determining the shock
309 train properties. A small error in resolving the boundary layer drastically changes the shape
310 of the leading shock, which influences the overall configuration of the shock train.

311 The difficulties in achieving grid-independent results reflects the characteristic of super-
312 sonic flows in long ducts being extremely complicated. The dependence of the shock train
313 on the grid size showed that as the grid is refined the differences between two subsequent
314 grids become gradually smaller leading to the conclusion that a finer grid is expected to give
315 results very close to Grid 7. Of the three turbulence models employed only the $k-\omega$ Wilcox
316 closely matches the experimental pressure distribution confirming to be the most suitable
317 for capturing the shock train characteristics.

318 From the 2D and 3D results the boundary layer thickness influences the shock train
319 shape and location in the duct. At the duct centreline, the flow properties showed that the
320 first shock wave is captured with the same strength. However, in 3D the flow behind the

321 first shock is decelerated more strongly, which then causes a mismatching of the subsequent
322 shock waves composing the shock train. Although two-dimensional simulations qualitatively
323 resolve the mechanism of formation of the shock train in long ducts, the absence of the
324 sidewall effects limits the accuracy. A 3D domain is necessary for the comprehension of the
325 flow physics. However, the solving of the RANS equations with a mesh structure composed
326 of a large number of cells requires an onerous computational power. This study has proven
327 that a compromise between an accurate solution and numerical resources is necessary and
328 that a 2D computation is not adequate to describe the characteristics of shock trains.

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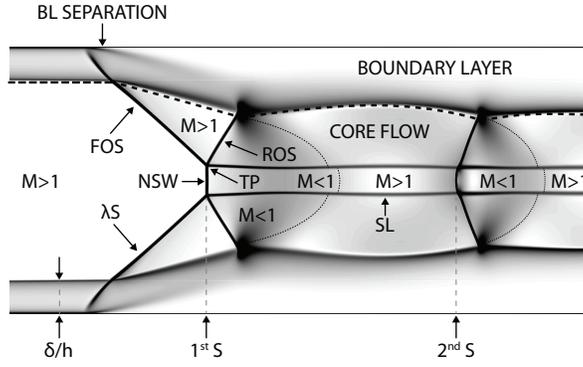


Figure 1. Schematic of the shock wave/boundary layer interaction in shock train obtained with the numerical approach used in the current study.

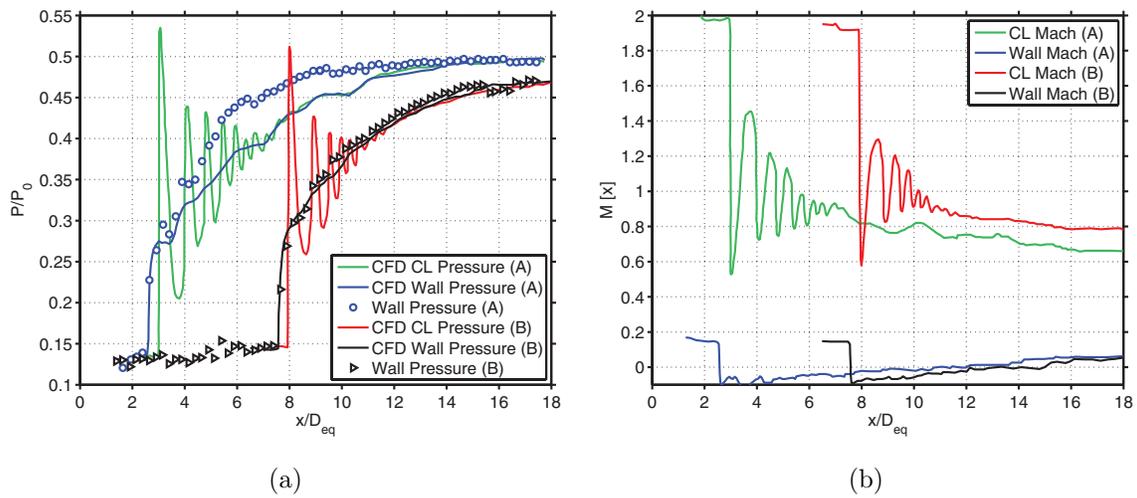


Figure 2. Numerical and experimental static pressure (a) and centreline Mach number (b) distributions obtained by Sun et al.²⁰ for different back-pressures.

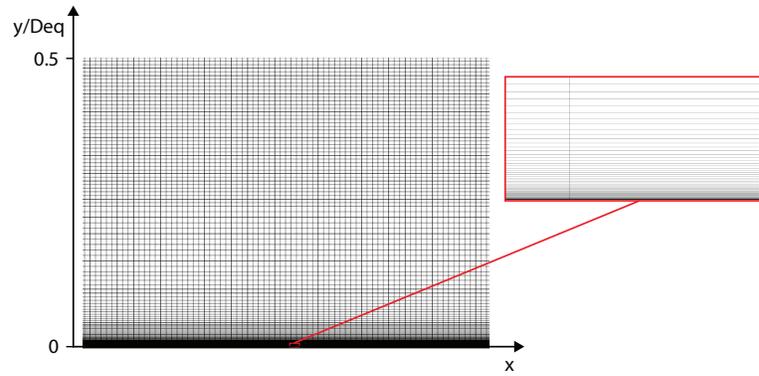


Figure 3. Portion of the half duct numerical grid employed in the 2D computational domain.

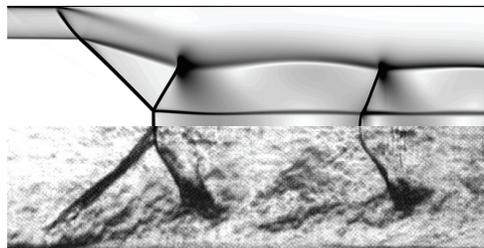
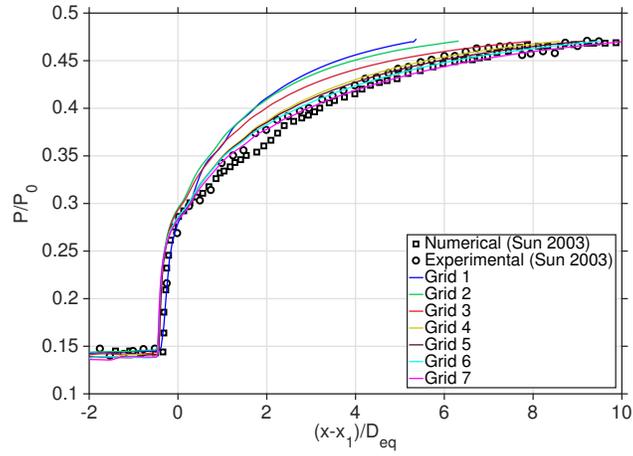
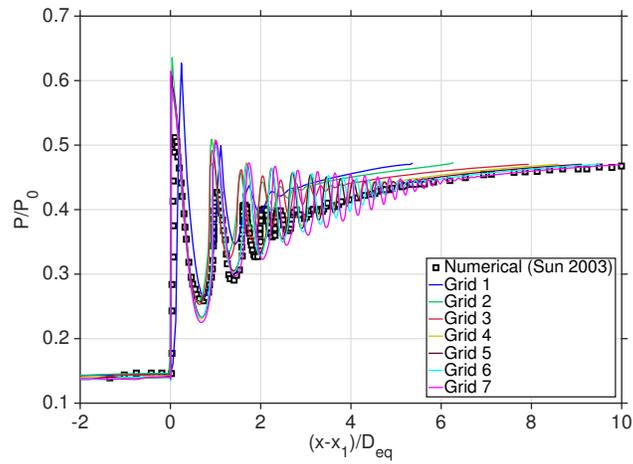


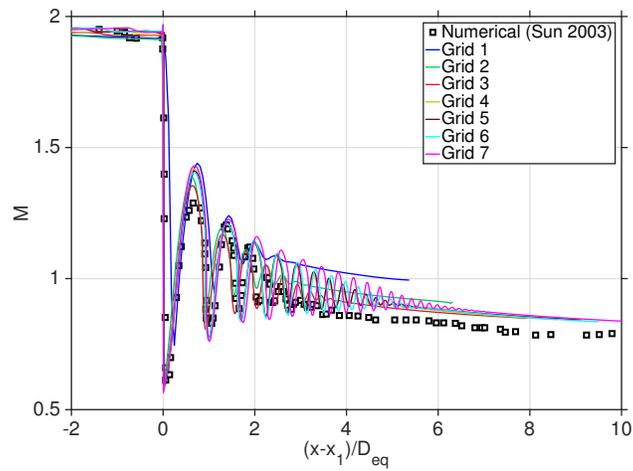
Figure 4. Comparison of schlieren photography from the reference results¹³ and the numerical density gradient magnitude obtained with the current numerical approach and .



(a)



(b)



(c)

Figure 5. Effect of grid size on the accuracy of pressure and Mach number distributions. a) Wall static pressure; b) Centreline static pressure; c) Centreline Mach number.

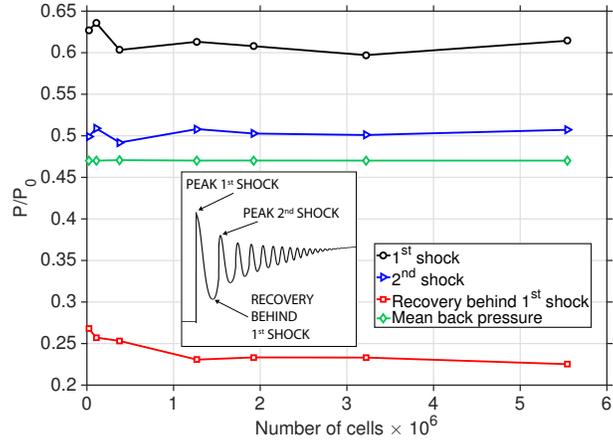


Figure 6. Variation of the value of pressure of different parts of the shock train with grid resolution.

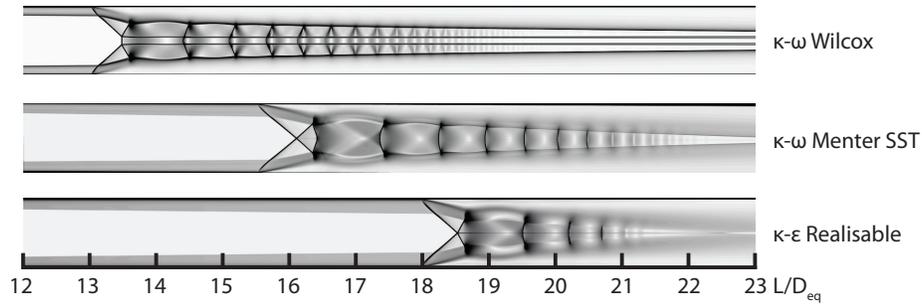


Figure 7. Numerical schlieren with different turbulence model.

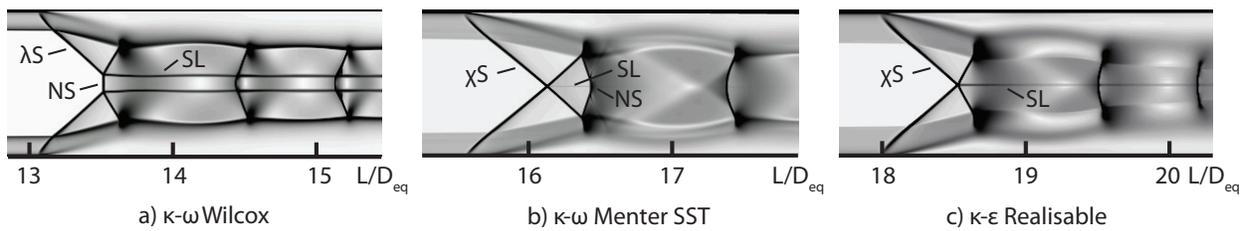


Figure 8. Close up of numerical schlieren at the corresponding first shock with different turbulence model.

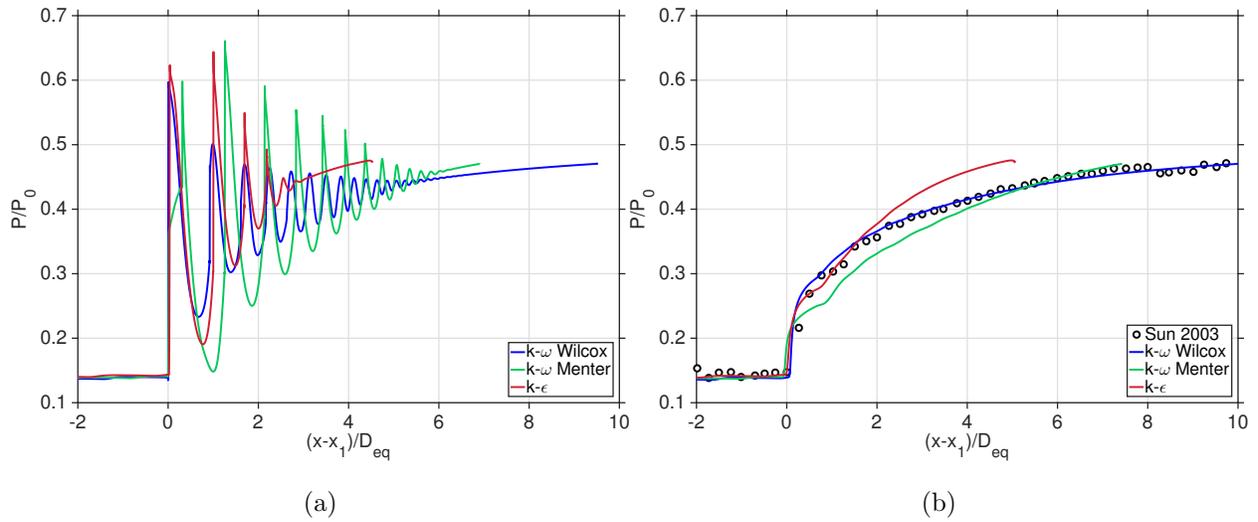


Figure 9. Effect of turbulence model on the accuracy of the static pressure distribution at the duct centreline (a) and at the wall (b). The plots are shifted for common pressure rise and normalised to the equivalent hydraulic diameter.

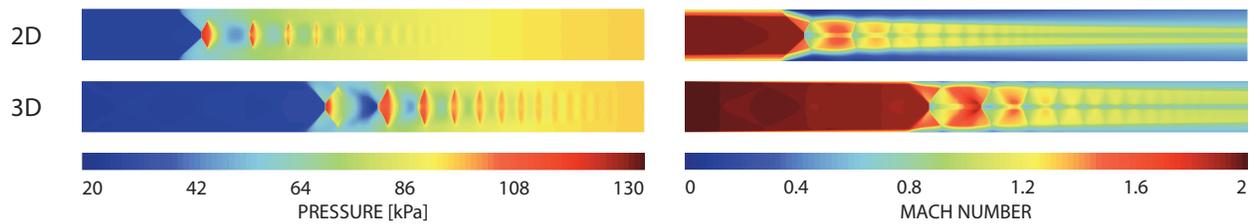


Figure 10. Comparison of pressure and Mach number contour in the 2D (upper) and 3D (lower) domains.

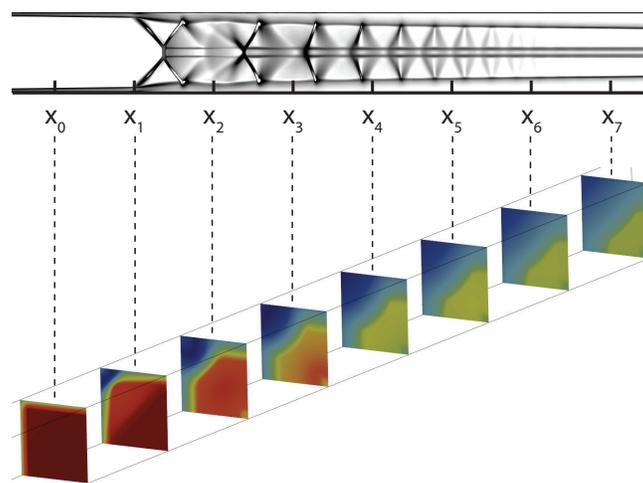


Figure 11. Numerical schlieren and Mach number contour at different axial locations.

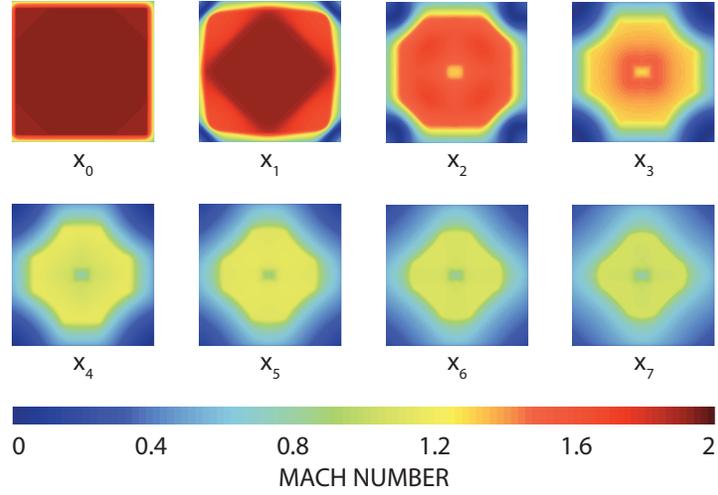


Figure 12. Mach number contour at different cross sections.

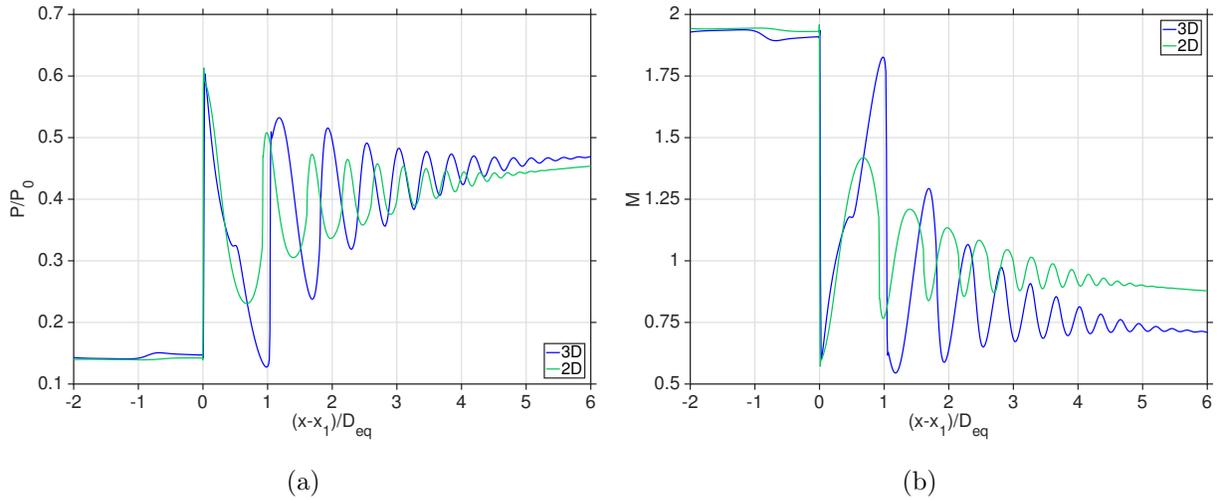


Figure 13. Centreline static pressure (a) and Mach number (b) distributions with 2D and 3D domain.